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1. Please correct the author's name on the abstract cards, located in the back of NAVORD Report 6683, to read as

James E. Danberg

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Aerodynamics Research Report No. 67

MEASUREMENT OF THE CHARACTERISTICS OF THE COMPRESSIBLE  
TURBULENT BOUNDARY LAYER WITH AIR INJECTION

Prepared by:

James E. Danberg

**ABSTRACT:** The compressible turbulent boundary layer on a porous flat plate with air injection has been studied experimentally in the Naval Ordnance Laboratory hypersonic wind tunnel. Boundary-layer velocity and temperature profiles were obtained at a Mach number of 5.1, a Reynolds number of  $1.5 \times 10^7$  per meter, wall to free-stream temperature ratios of 4.2 and 5.0, and rates of air injection from zero to 0.08 percent of free-stream mass flow. Experimental values of skin friction were obtained from the velocity profiles and the results were compared with the theories of Rubesin and Persh. Neither theory predicts the experimental data on an absolute basis but Rubesin does describe the trend within  $\pm 25$  percent. The measured total temperature and velocity profiles compared favorably with the profiles assumed in Rubesin's theory. However, the Rubesin expression for the  $u^+$  velocity at the interface between the laminar sublayer and the outer turbulent layer disagrees with experiment. The measurements indicate  $u^+$  values of  $14 \pm 3$ , independent of both wall temperature and injection rate.

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WHITE OAK, MARYLAND

3 September 1959

This report presents the results of the first phase of a comprehensive program intended to extend the basic knowledge of the effect of mass transfer on hypersonic turbulent boundary layers. Knowledge concerning the characteristics of mass-transfer cooling has considerable current interest because of its potential applications in alleviating intense aerodynamic heating. Two of these applications are, first, the protection of missiles and space vehicles during atmosphere re-entry and, second, the protection of high performance rocket nozzles.

The next phase of this program will extend the range of the parameters studied, with particular attention to heat-transfer measurements, location of transition, and details of the profile near the sublayer.

This work was jointly sponsored by the U. S. Naval Bureau of Ordnance, the U. S. Air Force, the U. S. Army, and the Re-entry Body Section of the Special Projects Office, Bureau of Ordnance, under the Applied Research Program in Aeroballistics and Materials. It was carried out under Task Numbers NOL 502-335/51014/01, NOL 291, NOL 300, and NOL 363.

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**SYMBOLS**

$C_f$	skin-friction coefficient = $2\tau_w/\rho_\infty u_\infty^2$
$C_q$	nondimensional injection rate = $\rho_w v_w/\rho_\infty u_\infty$
$K$	constant = 0.4
$l/n$	exponent in the velocity profile
$M$	Mach number
$p$	pressure, mmHg
$Pr$	Prandtl number
$Re$	Reynolds number
$r$	recovery factor
$T$	Temperature, °K
$u$	velocity parallel to the surface, m/sec
$u^+$	nondimensional velocity = $u \sqrt{\rho_w/\tau_w}$
$v$	velocity of injected air normal to the surface, m/sec
$y$	distance normal to the surface, mm.
$y^+$	nondimensional distance from the wall = $y \sqrt{\tau_w \rho_w}/\mu_w$
$\delta$	boundary-layer thickness, mm.
$\delta^*$	boundary-layer displacement thickness, mm.
$\theta$	boundary-layer momentum thickness, mm.
$\mu$	coefficient of viscosity, Kg/m-sec
$\rho$	density, Kg/m <sup>3</sup>
$\tau$	shear stress, Kg/m-sec <sup>2</sup>

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SYMBOLS

**Subscripts**

- a an arbitrary point on the velocity profile in the turbulent outer layer
- i interface
- m measured or indicated
- w surface
- x distance from the leading edge
- $\infty$  free stream
- e adiabatic wall temperature
- 0 supply conditions
- 2 condition behind normal shock

## INTRODUCTION

1. The high speed re-entry into the atmosphere of ballistic missiles and space vehicles requires the development of techniques for protecting their payload against the intense heating produced. One promising system is mass-transfer cooling. In fact, some nose cone materials ablate during re-entry and ablation under these conditions is a form of mass transfer cooling. For many years mass transfer techniques have been employed by turbo-jet and rocket engine manufacturers in protecting components exposed to high temperature combustion products. The missile application, however, differs from the engine application in the conditions outside the boundary layer.
2. Interest in this field has produced literature of considerable extent. Of particular interest are the incompressible turbulent boundary-layer measurements of Mickley, et al., (reference (a)) and the compressible laminar and turbulent boundary-layer measurements on porous cones and flat plates of Leadon and Scott (reference (b)) and Rubesin, Pappas, and Okuno (references (c) and (d)).
3. In order to increase the basic understanding and to extend the experimental data to higher speeds, the Naval Ordnance Laboratory has undertaken an extensive experimental program to study the hypersonic turbulent boundary layers with mass (gas) injection. The experiments reported herein represent only the first phase of this program and are a logical extension of the solid wall turbulent boundary-layer studies of reference (e).
4. This first phase of the program is concerned with measuring the pressure and temperature profiles on a porous flat plate at  $M_{\infty} = 5.1$ , a Reynolds number of  $1.5 \times 10^7$  per meter,  $T_w/T_{\infty}$  of 4.2 and 5.0, and for rates of air injection from zero to 0.08 percent of the free-stream mass flow. From these profiles, values of skin-friction and other boundary-layer parameters were obtained.

## Experimental Equipment and Techniques

5. The experiments described in this report have been conducted in the NOL Hypersonic Tunnel No. 4, the basic components of which are described in references (f) and (g). This wind tunnel is capable of operating continuously at Mach numbers from 5 to 10 with supply temperatures from 300°K to 800°K and supply pressures from 1 to 50 atmospheres. During

this particular experiment a Mach number 5.1 uniform-flow-nozzle was used with constant supply conditions of 380°K and 3 atmospheres.

6. The porous plate (Figure 1 and Figure 2) consisted of a flat plate (60.96x15.87 cm.) with a sharp leading edge (10° angle) and containing a sintered, woven, stainless-steel, wire-mesh insert (48.90x12.70x0.952 cm.). The porous material (manufactured by Aircraft Porous Media, Inc.) starts 6.98 cm. from the leading edge. The exposed side consisted of .127 mm. wire and was flat within a few tenths of a millimeter.

7. The porous material was tested for uniformity of mass flow with a venturi-tube flow meter. The calibration tests were performed at the correct operating pressures and temperatures on both sides of the porous material, thereby simulating the viscous and compressibility effects within the plate that are present during the actual wind-tunnel tests. The results showed the insert did not have the specified uniform mass-flow distribution. In two small regions near the center of the plate the mass flow/sq. cm. was twice the average while near the edges of the plate no flow was detected. The exact effect on the experiment of this non-uniformity is unknown now. However, the locations at which the boundary layer was investigated were more than 26 cm. down stream from the start of the porous insert and thus were influenced, to some extent, by about half the total injected mass.

8. The porous insert had 8 thermocouples cemented in the surface (Figure 1 and Figure 2). Six grooves 2.51 mm wide by .25 mm deep were machined in the exposed surface from the edge to the centerline of the plate and two others were placed diagonally so as to have at the 34.4 cm station three thermocouples, one on the centerline and two 1.27 and 3.31 cm from the centerline respectively. These provided a measurement of surface temperature and a check on its uniformity. The underside of the porous material was grooved and contained 15 U-shaped copper tubes. Silicon oil (Dow Corning DC-200, 2 centistoke) was circulated through these tubes entering at the downstream end of the plate passing up through the chamber beneath the plate and then back through the portion of each tube imbedded in the porous material. A view of the underside of the plate is shown in Figure 3. This system provided good temperature control and was necessary because the injected air was exposed to the hot back cover of the plate. The inside of the back cover forms the plenum chamber for the injected air. The temperature of the coolant and of the injected air was held constant by circulating them through separate copper coils immersed in a thermostatically controlled bath of either dry ice in alcohol or circulating tap water.

9. The boundary-layer measurements were made at two locations on the porous plate. One was 29.25 cm. and the other was 36.85 cm. from the leading edge. Previous results obtained with a solid flat plate (reference (e)) indicated that for the conditions of the present test, a fully developed turbulent boundary layer should exist at the two measuring stations.

10. At each station and for two essentially constant wall temperatures, namely,  $T_w/T_\infty = 4.23$  and  $5.04$ , the boundary layer was surveyed without injection and with four rates of air injection. The four injection rates are  $\rho_w v_w / \rho_\infty u_\infty$

$4.24 \times 10^{-4}$ ,  $6.18 \times 10^{-4}$ , and  $7.88 \times 10^{-4}$ . The injection rates are based on the total injected mass flowing through the porous insert as measured by a flowmeter in the air supply line divided by the surface area of the plate. The injection rate is made dimensionless by dividing by the free-stream mass flow per unit area,  $\rho_\infty u_\infty$ .

11. Pitot pressure probes and total temperature probes were used to survey the boundary layer at both stations. In addition, static probe measurements were taken in the free-stream and for a short distance into the boundary layer. The Pitot probe and total temperature probe were essentially of the same external geometry. Both were made from thin-walled hypodermic tubing with a rectangular opening of half height .152 mm. The pressures were measured on either a mercury manometer with a range of 0 to 1 atmosphere absolute pressure or on a silicon oil manometer with a range of up to 35 mmHg. absolute. The reading accuracies of the two manometers were  $\pm 0.1$  mmHg. and  $\pm 0.001$  mmHg. respectively.

12. The internal construction of the total temperature probes was basically the same as the probes described in reference (f). In the present case, the probe consists of a .127 mm. iron-constantan thermocouple within a vented stainless-steel tube which serves as a radiation shield and as a device to establish known and constant flow conditions around the thermocouple junction. The latter is essential for obtaining a unique calibration in terms of  $(T_m - T_\infty)/(T_0 - T_\infty)$  as a function of the parameter  $p_2/T_m^{7/4}$  (reference (f)). The probe was calibrated in the wind-tunnel free stream by varying the supply pressure and temperature so that the parameter  $p_2/T_m^{7/4}$  was varied over the range encountered in the boundary-layer surveys. From this information, the local total temperature was

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calculated from the indicated temperature, the local Mach number, and the Pitot pressure.

13. The above probes were supported in the tunnel by a traversing mechanism which allowed positioning of the probe with an accuracy of .025 mm. in a plane perpendicular to the centerline of the plate and extending from the plate surface about 7.62 cm. Each traverse was made from the free stream to the plate, the location of which was indicated by the completing of an electric circuit when the probe first touched the surface.

## RESULTS

### Boundary-Layer Profiles

14. The measured boundary-layer data are given in Tables 1 through 19. The Pitot pressures have been converted to Mach number by assuming constant static pressure through the boundary layer and using the standard flow tables for air, reference (1). The static temperature has been calculated from the probe indicated temperature with the use of the static pressure, local Mach number and by reference to the probe calibration curve. The velocity was calculated from this data using the standard flow equations.

15. Figures 4 through 11 contain the graphical representation of this tabulated data. The velocity distribution through the boundary layer (Figures 4 through 7) for zero blowing shows the typical, fully developed, turbulent profile. The air injection decreases the velocity more or less uniformly over a considerable part of the boundary layer and it also increases the total thickness. The distortion of the shape of the profile, due to the addition of air, occurs mostly at the point of greatest rate of change of curvature. At that point, the curves become flatter. The effect on the profile of changing the wall temperature and Reynolds number is difficult to detect with the limited data. There is evidence, however, that a decrease of the wall temperature produces the same sort of flattening near the knee of the curve as described above.

16. The general appearance of the total and static temperature distributions through the boundary layer (Figures 8 through 11), in the zero injection cases, is also that of fully developed turbulent profiles. Injection of air into the boundary layer decreases the total temperature in a manner

similar to the effect of injection on the velocity profile. The static temperature, however, increases with blowing rate.

#### Boundary-Layer Parameters

17. Certain properties of the boundary layer have been deduced from these profiles, such as: (Table 20)

$\delta$ , the total boundary-layer thickness  
 $\delta^*$ , the displacement thickness

$$\delta^* = \int_0^\delta \left(1 - \frac{\rho u}{\rho_\infty u_\infty}\right) dy \quad (1)$$

$\theta$ , the momentum thickness

$$\theta = \int_0^\delta \frac{\rho u}{\rho_\infty u_\infty} \left(1 - \frac{u}{u_\infty}\right) dy \quad (2)$$

where  $\delta^*$  and  $\theta$  were obtained from graphical integration of the profile data.

18. Each of these parameters increases roughly linearly with injection rate (Table 20). A decrease in wall temperature generally decreases slightly each thickness parameter, but the variation in wall temperature was too small for a quantitative statement.

19. The value of the ratio  $\delta^*/\theta$ , based upon these measurements, is between 10.5 and 11.5, independent of the injection rate.

20.  $n$ , the inverse of the velocity profile exponent in the equation

$$\frac{u}{u_\infty} = \left(\frac{y}{\delta}\right)^{1/n} \quad (3)$$

was obtained from a log-log plot of the velocity against  $y$ . As might be expected,  $n$  decreases rapidly and nearly linearly with the injection rate. In fact, at  $Cq=8 \times 10^{-4}$ ,  $n$  has approximately one half of its zero injection value. The zero injection cases where  $n$  is  $9 \pm .5$  are consistent with the impermeable wall data of reference (e), but are higher than the nozzle wall measurement of reference (j) ( $5 \leq n \leq 7$ ).

## Local Skin Friction

21. The skin friction was calculated in this experiment by extrapolating the velocity profile to the wall. There is an important difficulty in this procedure when there is blowing through the wall. That is, it is not necessary for the velocity to be zero at the wall. Since the wall is porous it is possible, on the average, for the injected air velocity to have a component along the plate. Since the shear stress can penetrate the pores, it is also possible that the velocity component has a preferred direction, that is down stream. Thus, whereas with an impermeable surface the velocity is interpolated, in the case of a porous surface it is an extrapolation. The present data do not necessarily indicate a zero velocity at the wall. Thus, the specific curve and the resulting skin-friction coefficient

$$C_f = 2\mu_w \left( \frac{\partial u}{\partial y_w} \right) / \rho_\infty u_\infty^2 \quad (4)$$

depend strongly on the individual drawing the curve. Figures 12 through 15 show details of the last few measured points, i.e., in the region up to 1 mm from the wall. The skin-friction coefficients,  $C_f$ , have been calculated by the above formula and are based on a tentative extrapolation, as shown in Figures 12 through 15. If the velocity is forced to go through zero at  $y=0$ , the skin friction would be increased on the average by 44 percent, or in some cases, up to 100 percent.

22. As stated before, a fully developed turbulent boundary layer should be expected at about 21.6 cm. from the leading edge based on reference (e) results and from that point the local skin friction should monotonically decrease with increasing distance. The present results with  $u_w \neq 0$  at the 29.25 cm. station do not show this decrease. With zero injection, the skin-friction coefficient is less than at the 36.85 cm. station. In fact, the  $C_f$  at 29.25 cm. is 30 percent lower than obtained under similar conditions on the solid flat plate. The present results might be interpreted as evidence of a later onset of transition or as a more extended transition region on the porous plate. On the other hand, if the velocity extrapolation is changed to give  $u=0$  at the wall the corresponding  $C_f$  is increased. The 29.25 cm. station is then in better agreement with the previous solid flat-plate results but the 36.85 cm. station is considerably higher.

## COMPARISON WITH THEORY

## Skin Friction

23. Results in terms of  $C_f$  with  $u_w \neq 0$  are shown in Figures 16 and 17, and included are the  $C_f$  calculations from the analytical approaches of Rubesin (reference (k)) and Persh (reference (l)). For the downstream station, the zero injection data are fairly closely predicted by both analyses. The upstream zero injection cases and all the blowing cases are as much as 50 percent below Rubesin's calculation. Persh's theory is also higher than the data for the upstream station but for  $C_q$  greater than  $4.2 \times 10^{-4}$ , the calculated values fall below the data. In order to demonstrate more clearly the trend of the experimental data with blowing rate, the results in terms of  $C_f$  to  $C_f$  without injection are plotted in Figure 18. The bands marked "theoretical curve" were obtained from references (k) and (l) by evaluating  $C_f$  for the  $Re_\theta$ ,  $n$ ,  $T_w/T_\infty$  values associated with the experimental points. Each set of data exhibits a consistent decrease in  $C_f$  ratio with increasing  $C_q$ . Some of the data agree quite well with Rubesin's prediction.

## Total Temperature Profiles

24. In addition to the direct comparison of skin-friction coefficient, the Crocco temperature-velocity relation as used in the analyses of references (k) and (l) has been checked against the experimental data. Usually, the Crocco equation is given in the form:

$$\frac{T}{T_\infty} = \frac{T_w}{T_\infty} + \left( \frac{T_e - T_w}{T_\infty} \right) \frac{u}{u_\infty} - \left( \frac{T_e - T_\infty}{T_\infty} \right) \left( \frac{u}{u_\infty} \right)^2 \quad (5)$$

where

$$T_e = r (T_{\infty} - T_\infty) + T_\infty \quad (6)$$

Both Rubesin and Persh, tentatively assume the recovery factor  $r$  invariant with injection rate. Rubesin used  $r=1$  and Persh assumes  $r=Pr^{1/3}$ , where the Prandtl number ( $Pr$ ) is evaluated at  $T = T_w$ . Using the experimental velocity distribution together with Equation (5) it was found that the experimental and computed temperature distributions generally agree within 5 to 8 percent. The momentum equation is, however, relatively insensitive to the temperature profile so that the computed temperature is probably acceptable in the analyses of references (k) and (l).

25. In order to obtain a more specific indication of the validity of the Crocco equation in describing the temperature-velocity relation in the turbulent boundary layer with air injection, a representation was chosen which tends to amplify the discrepancies. For this purpose Equation (5) was rewritten in the form

$$\frac{T_o - T_w}{T_{o_\infty} - T_w} = \frac{T_e - T_w}{T_{o_\infty} - T_w} \frac{u}{\bar{u}} + \frac{T_{o_\infty} - T_e}{T_{o_\infty} - T_w} \left( \frac{u}{\bar{u}_\infty} \right)^2 \quad (7)$$

Figures 19 and 20 show the comparison of the experimental data with Equation (7), using  $r=1$  and  $r=Pr^{1/3}$ . The computations were performed for selected values of the Mach number, wall and free-stream temperatures which are representative of the experimental test conditions, namely, an average Mach number  $M=5.08$ ,  $T_w/T_\infty = 4.23$  and 5.05, and  $T_\infty = 61.3^\circ K$ . The four figures clearly show that the Crocco equation does not describe all details of the measured temperature-velocity distribution and its variation with heat transfer. The discrepancies are most pronounced for the conditions of largest heat transfer,  $T_w/T_\infty = 4.23$ , Figures 19b and 20b. The deviations are up to 32 percent of  $T_{o_\infty} - T_w$ . In the outer turbulent portion  $0.8 \leq u/u_\infty \leq 1$ , the experimental data for each graph seem to define one curve, i.e., there is no noticeable effect due to the air injection. In addition, the data are fairly close to the Crocco prediction with  $r=Pr^{1/3}$ . When  $u/u_\infty < 0.8$  the agreement with the theoretical curves is poor.

#### Velocity Profiles

26. The comparison of the experimental boundary-layer velocity profiles with the profiles assumed by Rubesin and Persh provides another check of the theories. As previously stated, neither of the theories accurately describes the experimentally determined variation of skin-friction coefficient with air injection (Figures 16 and 17). While Persh's theory predicts fairly accurately the value of the skin-friction coefficient for zero injection rate, it overestimates the effect of air injection. Rubesin's theory, on the other hand, predicts relatively closely the trend of changing skin-friction coefficient with air injection, but the absolute values of  $C_f$  are too high.

27. The Rubesin analysis describes the boundary-layer

velocity profile by two equations, one for the turbulent outer layer and one for the sublayer, and by an empirically determined interface which separates the two regions. The pertinent equations from reference (k) when modified for a finite velocity at the wall are:

$$\text{Laminar sublayer} \quad \mu \frac{du}{dy} = \tau_w + \rho_w v_w (u - u_w) \quad (8)$$

$$\text{Turbulent outer layer} \quad \rho k^2 y^2 \left( \frac{du}{dy} \right)^2 = \tau_w + \rho_w v_w (u - u_w) \quad (9)$$

$$\text{Interface location} \quad u = u_i = 13.1 \sqrt{(\tau_w / \rho_w)(T_w / T_\infty)} \quad (10)$$

28. Equation (10) was found inadequate for correctly predicting the location of the interface. Applying Equation (10) to the present experimental data, the calculated  $u_i$  was found, in most cases, to exceed the free-stream velocity. Therefore, it is useless in the present comparison for predicting the extent of the laminar sublayer.

29. In order to find a relation for  $u_i$  which more realistically describes the variation of the interface location and to determine how well Equations (8) and (9) describe the velocity profile with heat transfer and mass injection, the following procedure has been adopted. Equations (8) and (9) for the velocity profiles were numerically integrated using experimental values for skin friction, injection rates, and temperature profiles. The integration in the laminar sublayer was started at the wall where  $y=0$  and  $u=u_w$ . In the turbulent case, the integration was started at a point on the experimental velocity profile judged to be well within the turbulent outer portion of the boundary layer. The intersection of the calculated profiles gives the interface velocity  $u_i$ .

30. The integration of Equation (8) results in the following:

$$\frac{u}{u_\infty} = \frac{u_w}{u_\infty} + \frac{C_f}{2C_q} \left[ e^{-\left( \frac{\rho_w v_w}{\mu_w} \int_0^y \frac{dy}{\mu} \right)} - 1 \right] \quad (11)$$

where  $u$  can be obtained by numerically performing the integration and the lower limit of integration is  $y=0$ ,  $u=u_w$ . There are two reasons why it is difficult to judge the adequacy of this equation to describe the profile. First, very few data points were measured within the region and second, the equation cannot predict the correct profile near the point of

intersection with the outer turbulent part since the region of transition from the laminar sublayer to the fully turbulent region (i.e., buffer layer) has been neglected. Nevertheless, the few points available which were assumed on the linear part of the velocity curve in determining the skin friction fall below the curve defined by the above equation.

31. Integration of Equation (9) gives

$$\frac{u}{u_{\infty}} = \frac{u_w}{u_{\infty}} + \frac{1}{C_q} \left\{ \frac{C_q}{K} \int_{y_a}^y \sqrt{\frac{\rho_{\infty}}{\rho} \frac{dy}{y}} + \sqrt{\frac{C_f + C_q}{2} \left( \frac{u_w}{u_{\infty}} - \frac{u}{u_{\infty}} \right)} \right\}^2 \quad (12)$$

where again the integration can be performed numerically and where the subscript "a" refers to the matching point which was tentatively taken as equal to  $y^+ \cong 50$ . The constant  $k$  was assumed equal to 0.4 the non-blowing incompressible value. The velocity profile based on Equation (12) agrees very closely with the experimental profile in the range  $30 \leq y^+ \leq 100$ . The deviation from the experiment is +5 percent at  $y^+ = 30$  and -5 percent at  $y^+ = 100$ . The deviation near the outer edge of the boundary layer is similar to the "velocity defect" found in incompressible turbulent boundary layers. Figure 21 illustrates these results with a typical set of  $u^+$  versus  $y^+$  profiles.

32. The non-dimensional velocities at the point of intersection i.e.,  $u_i^+$  are shown plotted in Figure 22 against  $T_w/T_{\infty}$ , the parameter suggested by Equation (10). No trend is indicated. The measurements when plotted against  $C_q$  show only the slightest dependence on blowing rate. See Figure 23.

#### SUMMARY AND CONCLUDING REMARKS

33. The compressible turbulent boundary layer on a porous flat plate with air injection has been studied experimentally in the Naval Ordnance Laboratory Hypersonic Tunnel No. 4. Velocity and temperature boundary-layer profiles were obtained at a Mach number of 5.1, Reynolds number of  $1.5 \times 10^7$  per meter,  $T_w/T_{\infty}$  of 4.2 and 5 and for rates of air injection from zero to 0.08 percent of free-stream mass flow ( $.0008 \rho_{\infty} u_{\infty}$ ). Experimental values of skin friction were obtained from the velocity profiles.

34. As expected, the boundary-layer parameters, thickness ( $\delta$ ), displacement thickness ( $\delta^*$ ), momentum thickness ( $\theta$ ), and the

velocity profile exponent ( $1/n$ ) increase with rate of injection and the skin friction ( $C_f$ ) decreases.

35. Comparison of the skin-friction coefficient with theories of Rubesin (reference (k)) and Persh (reference (l)) were made and it was found that both predicted the non-blowing  $C_f$  for the down-stream station. Rubesin's, however, predicted the trend with blowing within  $\pm 25$  percent, while Persh's theory was consistently lower than the experiment, with the maximum being 50 percent.

36. The total temperature profiles showed that the Crocco profile describes the experimental data within about 8 percent of  $T$  or  $T_o$ , but, a closer examination of the data shows certain discrepancies between the total temperature profile and the theoretical profile.

37. The theoretical velocity profile as used in Rubesin's analysis has been examined. The empirical formula for the velocity at the interface between the laminar sublayer and the turbulent outer portion does not properly describe the interface location. In fact, the formula indicates  $u_i$  exceeds the free-stream velocity. The same differential equations for the velocity profile used in Rubesin's analysis were integrated numerically and fitted to the experimental data. The trend of the outer turbulent portion seems adequately described by this method within  $\pm 5$  percent in the range  $30 \leq y^+ \leq 100$ . The resulting interface velocity  $u_i^+$  was about  $14 \pm 3$  independent of changes in  $T_w/T_\infty$  and showed a very slight dependence on  $C_q$ .

38. The above represents the first results of a larger program intended to extend the basic knowledge of the mass-transfer phenomena in a systematic way. The present results are too limited in the range of the variables investigated and more tests are needed to establish and extend the trends with injection rate, heat transfer, and Reynolds number. To accomplish this, an extensive program is planned incorporating needed improvements, as for example, a porous insert with a more uniform injection rate, instrumented for measuring local heat transfer and static pressure at the surface, a cooling system adequate for a wider range of controlled temperatures, and refined instrumentation to resolve the questions concerning the region near the surface. Ultimately, mass addition should be investigated at higher Mach numbers with various injected gases, and under pressure gradient conditions.

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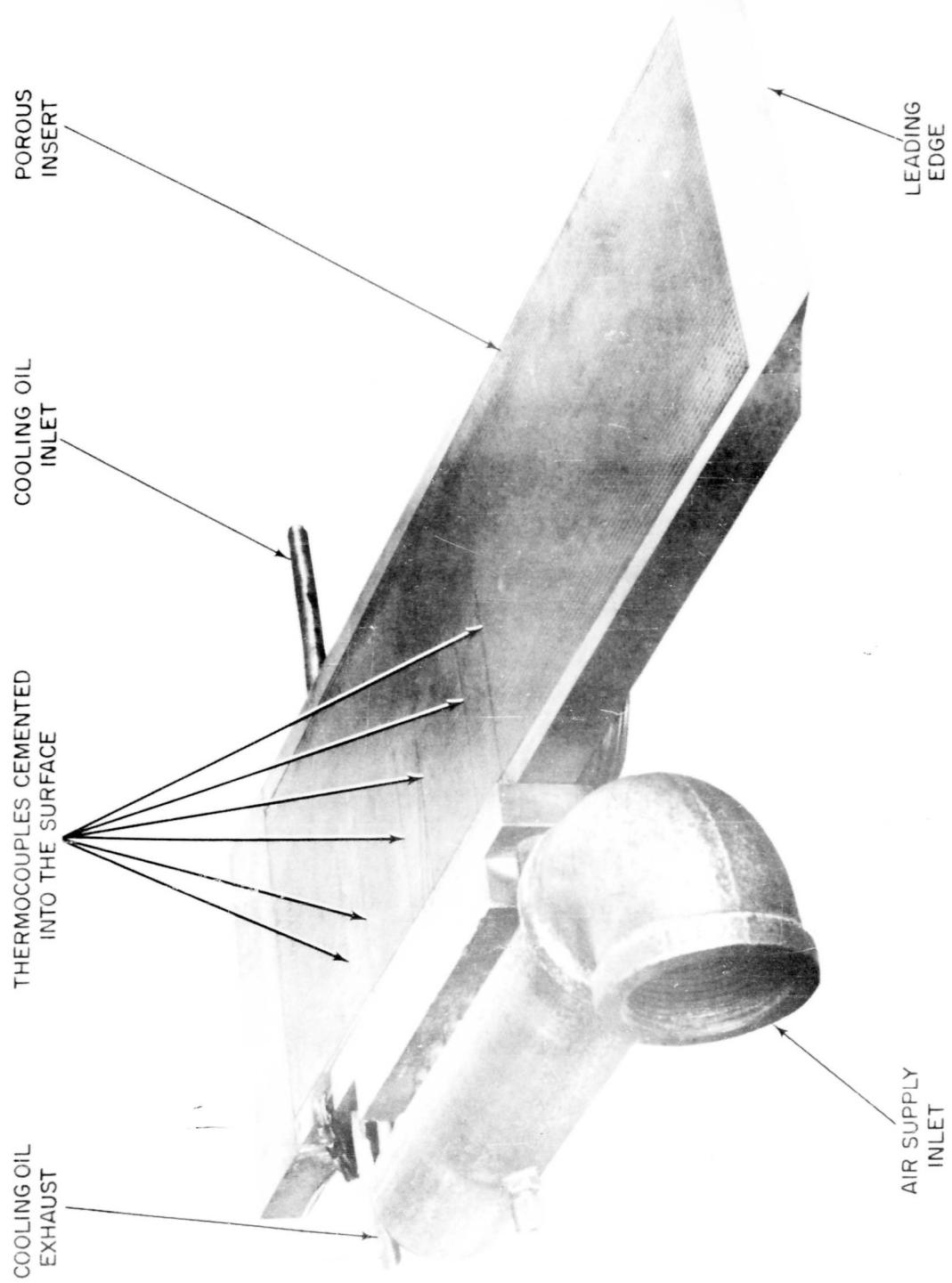
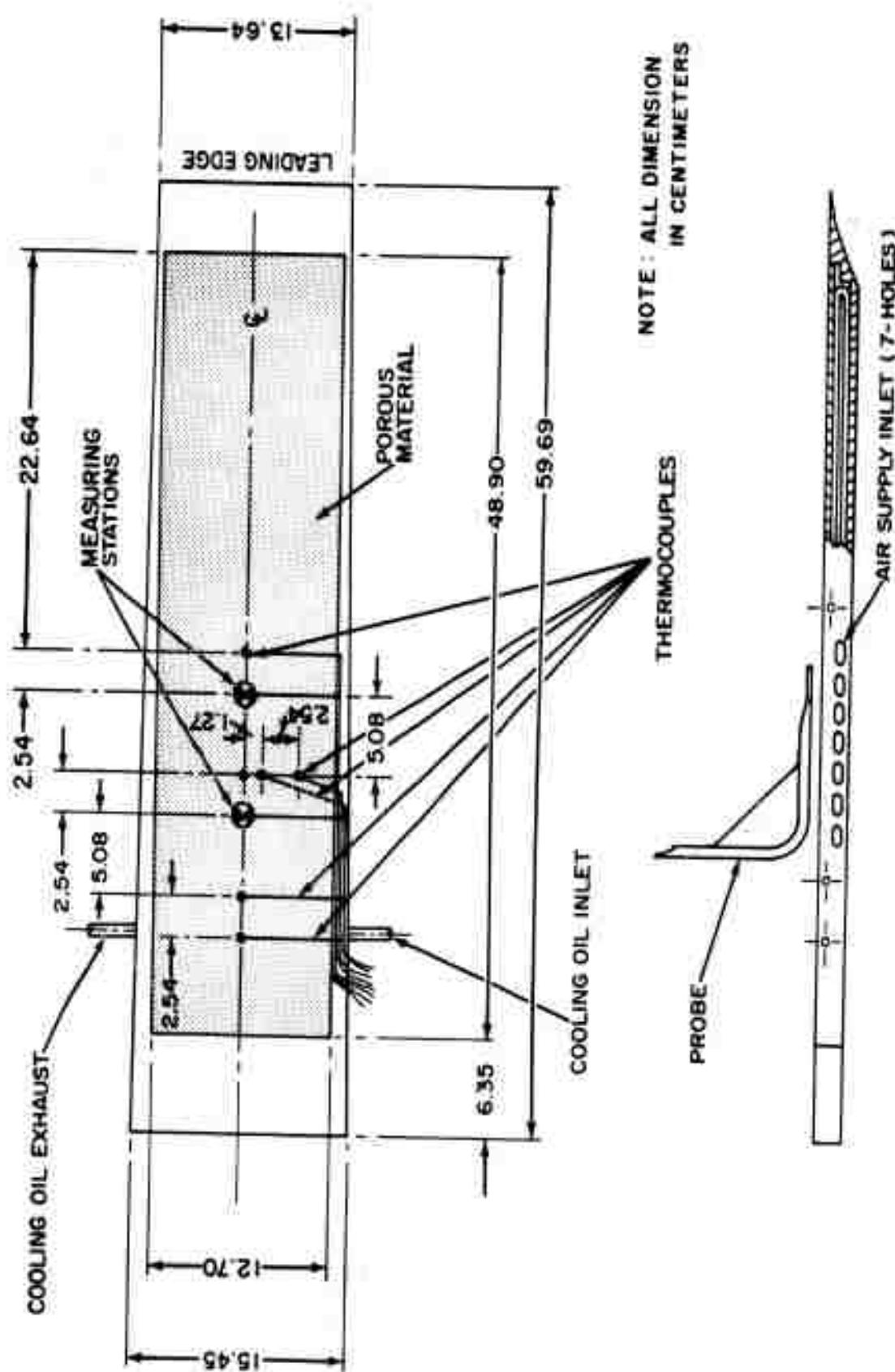


FIG. I TOP VIEW OF POROUS PLATE

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## FIG. 2 SKETCH OF MODEL

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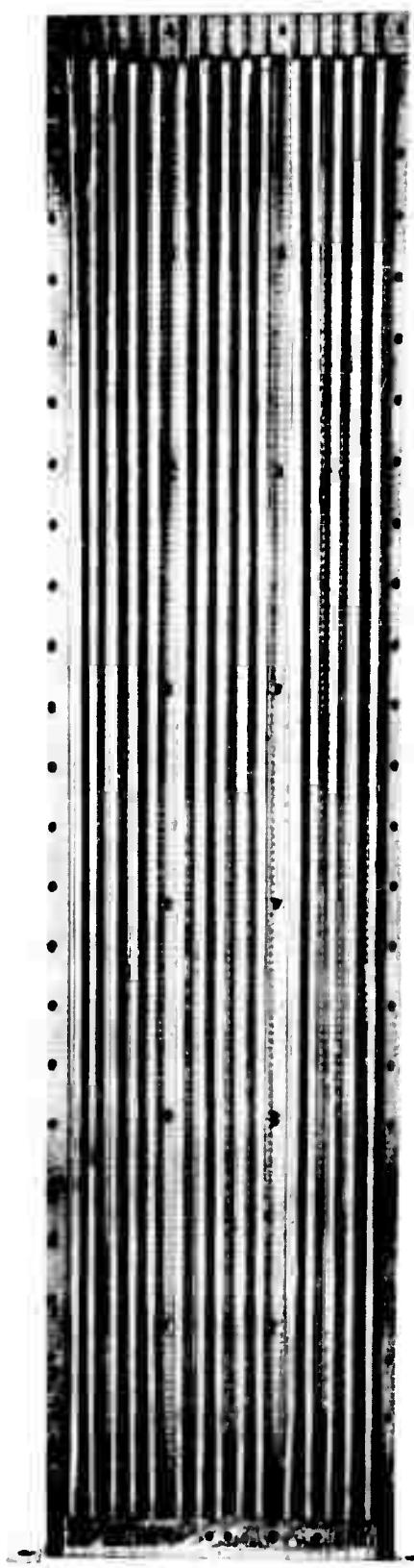
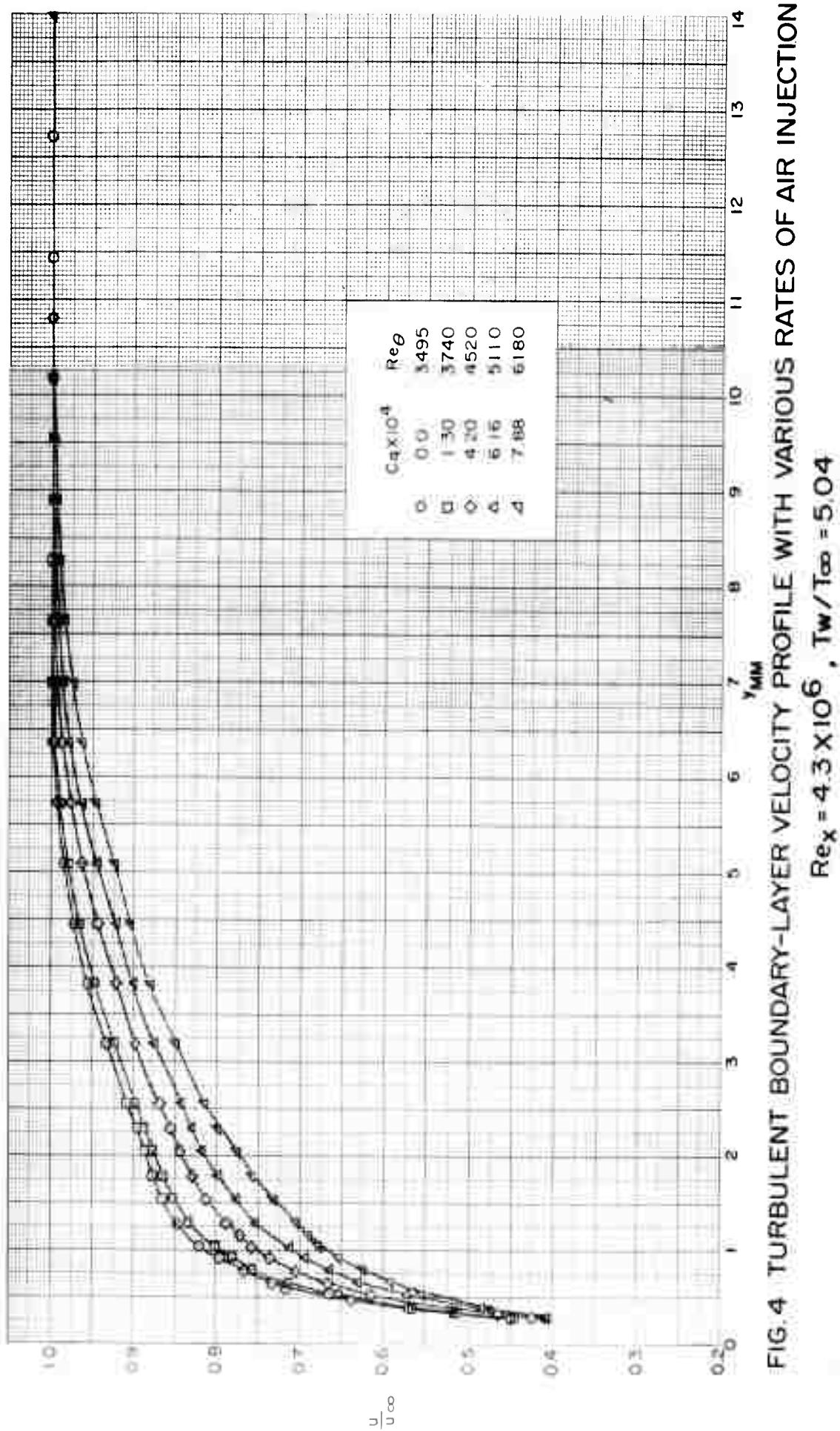


FIG. 3 UNDERSIDE OF POROUS INSERT SHOWING COOLING TUBES



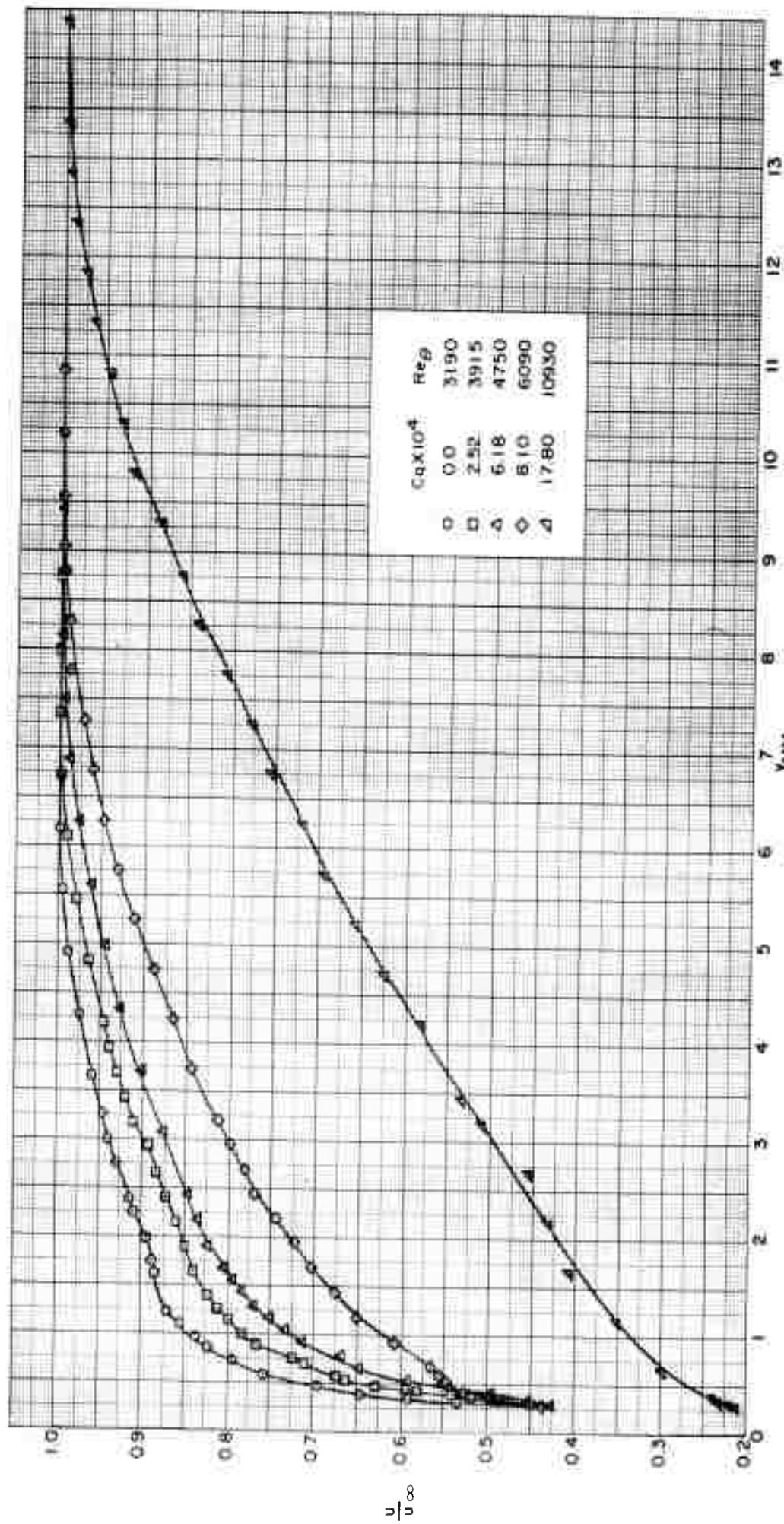
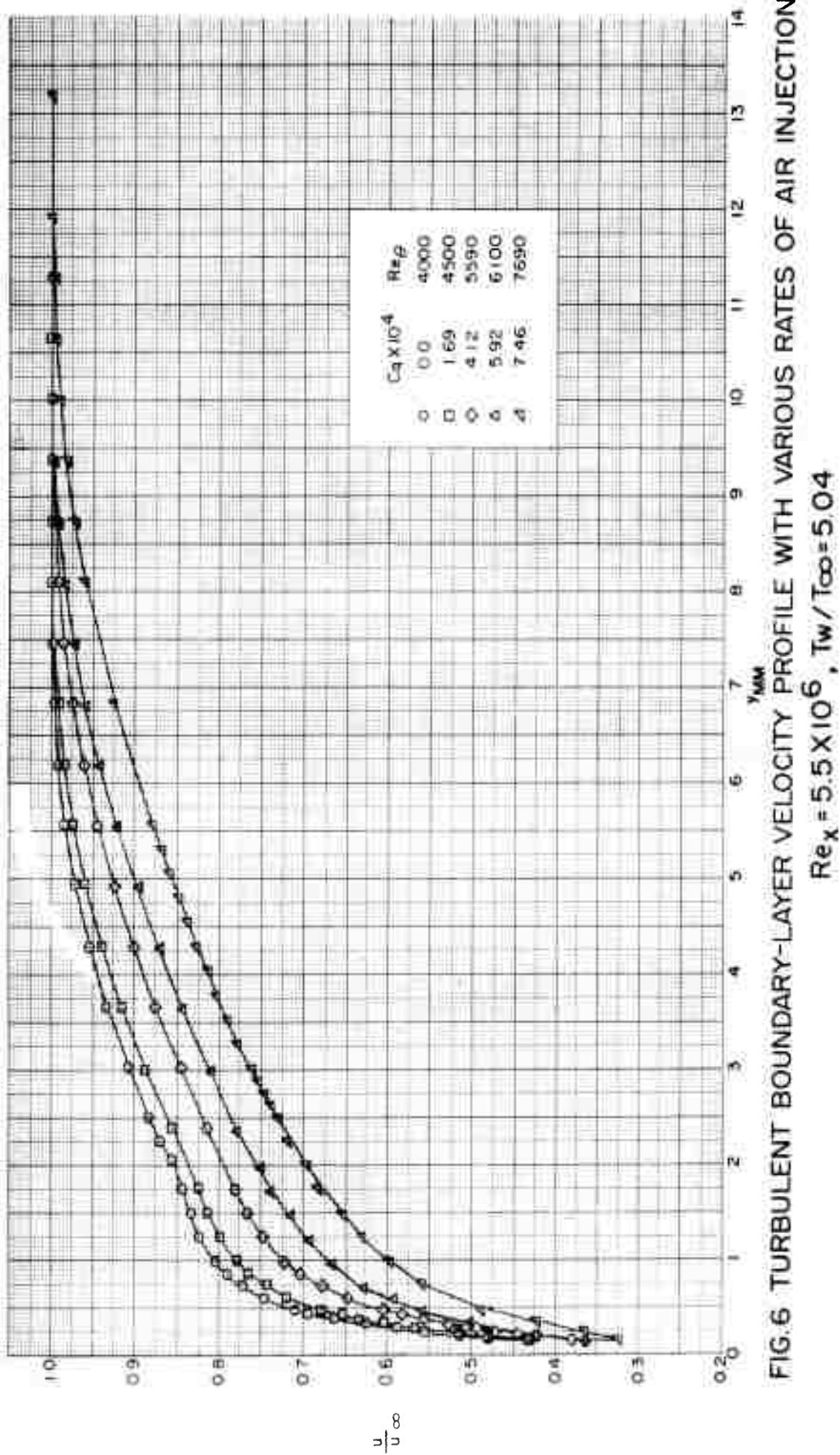


FIG. 5 TURBULENT BOUNDARY-LAYER VELOCITY PROFILE WITH VARIOUS RATES OF AIR INJECTION  
 $Re_x = 4.2 \times 10^6$ ,  $T_w/T_\infty = 4.23$



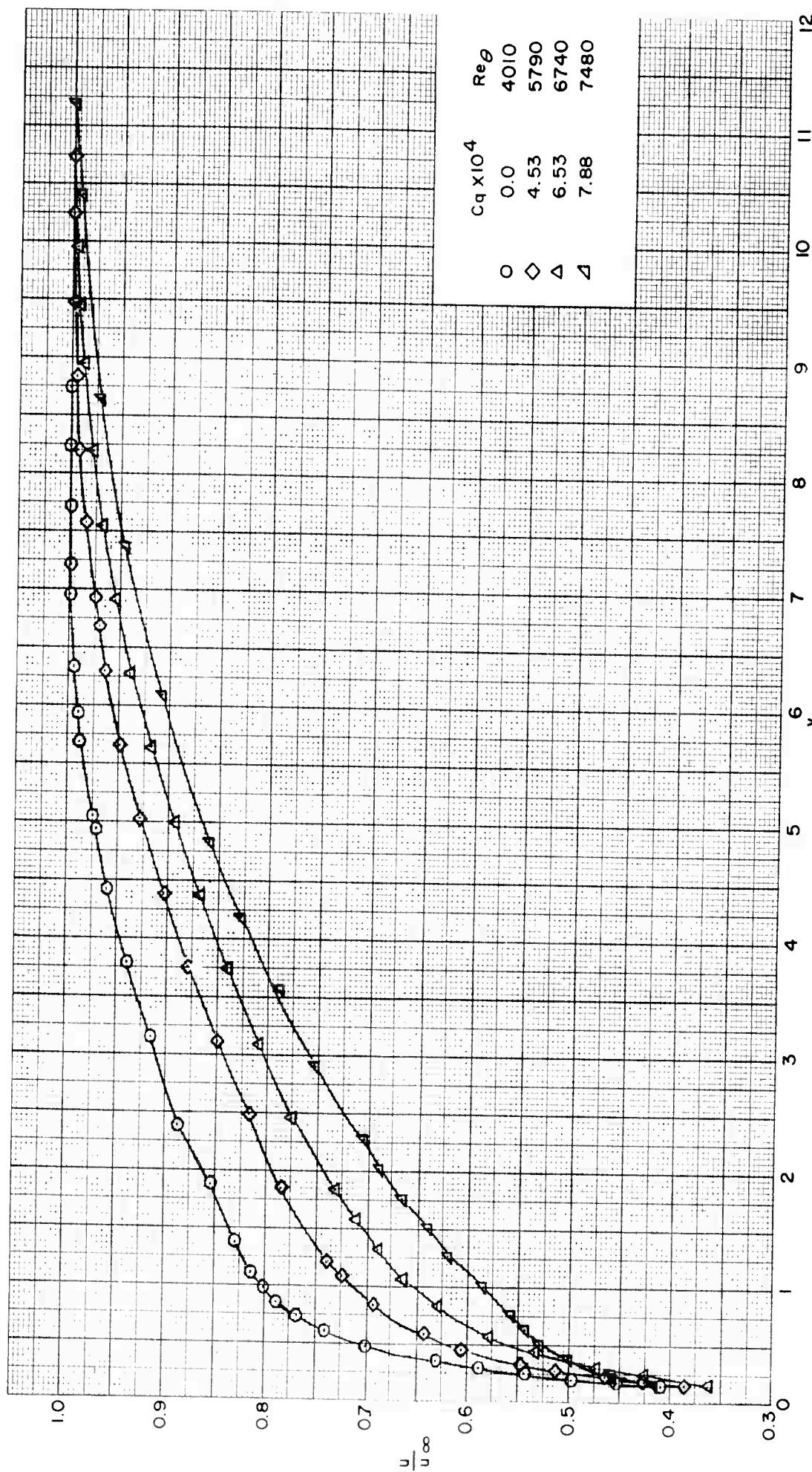


FIG. 7 TURBULENT BOUNDARY-LAYER VELOCITY PROFILE WITH VARIOUS RATES OF AIR INJECTION  
 $Re_x = 5.4 \times 10^6$     $T_w/T_\infty = 4.23$

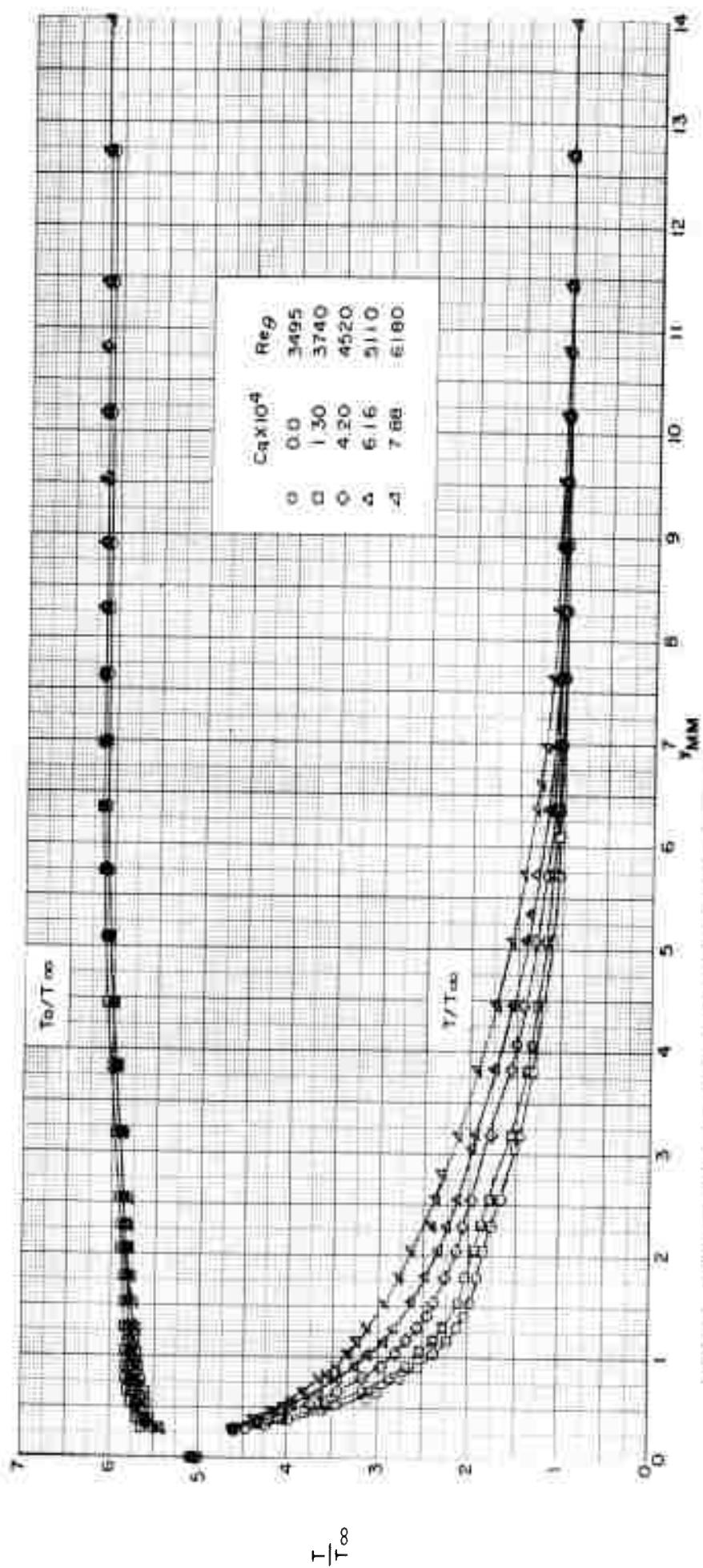


FIG. 8 TURBULENT BOUNDARY-LAYER TOTAL AND STATIC TEMPERATURE PROFILES  
WITH VARIOUS RATES OF AIR INJECTION  
 $Re_x = 4.3 \times 10^6$ ,  $T_w / T_\infty = 5.04$

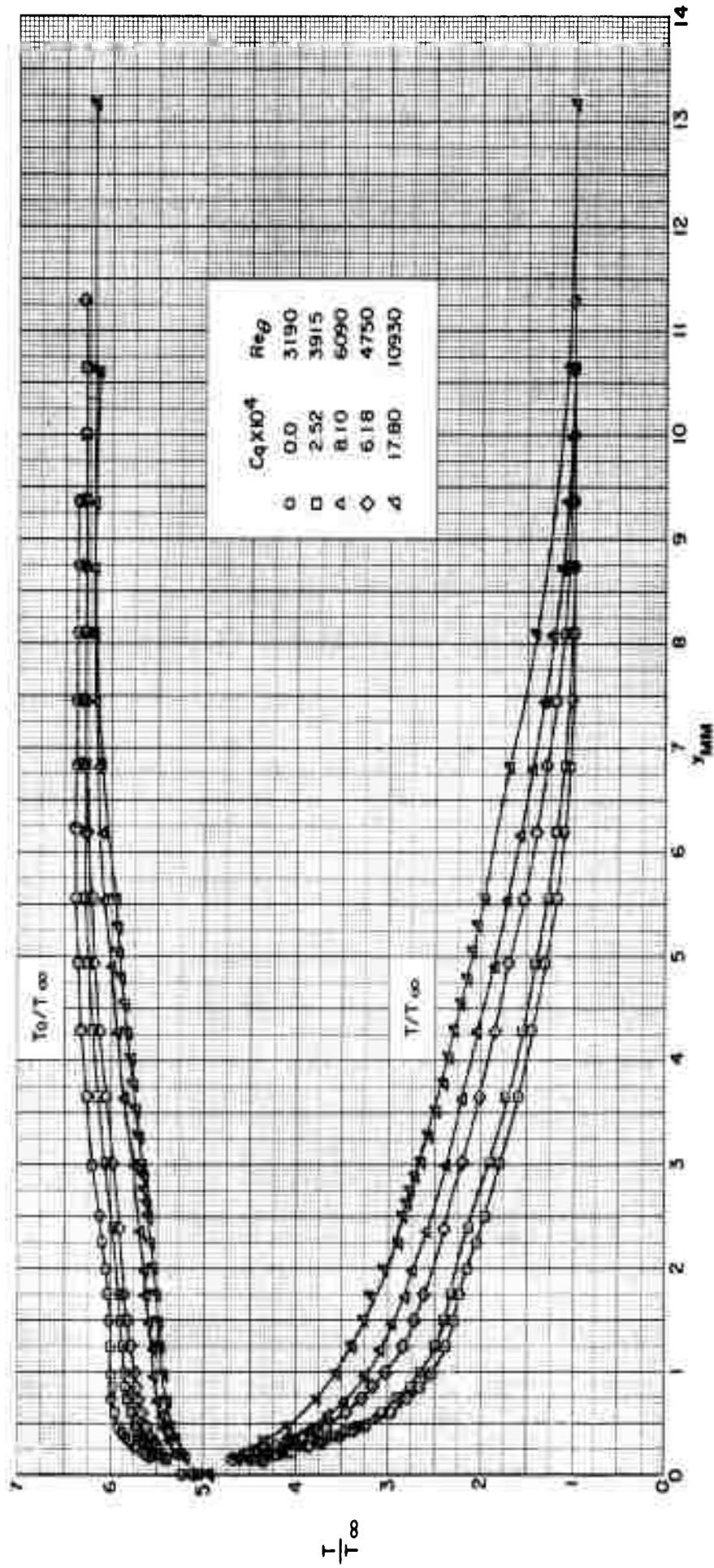


FIG. 9 TURBULENT BOUNDARY-LAYER TOTAL AND STATIC TEMPERATURE PROFILES  
WITH VARIOUS RATES OF AIR INJECTION  
 $Re_x = 4.2 \times 10^6$ ,  $T_w/T_\infty = 4.23$

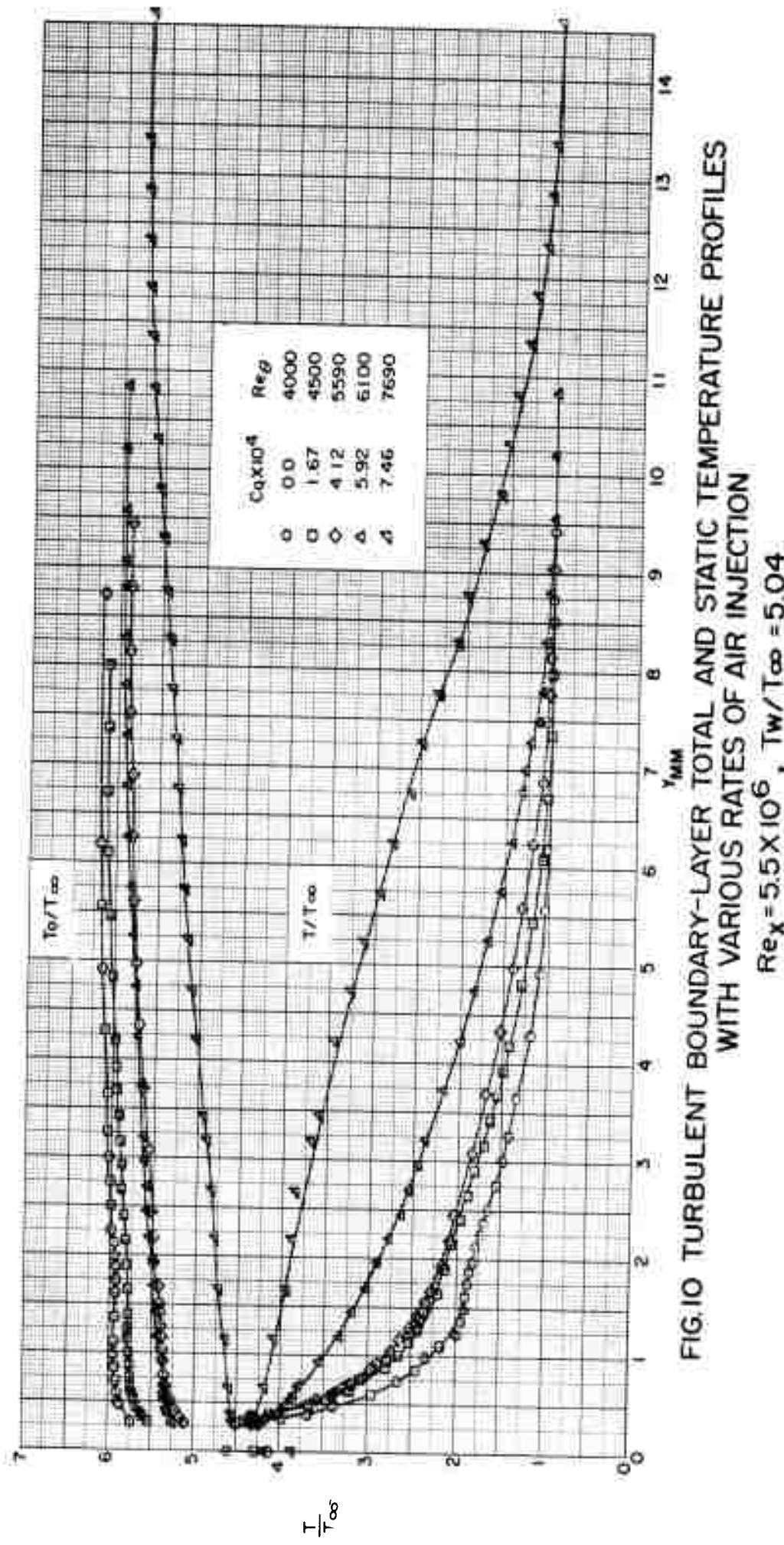


FIG. 10 TURBULENT BOUNDARY-LAYER TOTAL AND STATIC TEMPERATURE PROFILES  
WITH VARIOUS RATES OF AIR INJECTION  
 $Re_x = 5.5 \times 10^6$ ,  $T_w/T_\infty = 5.04$

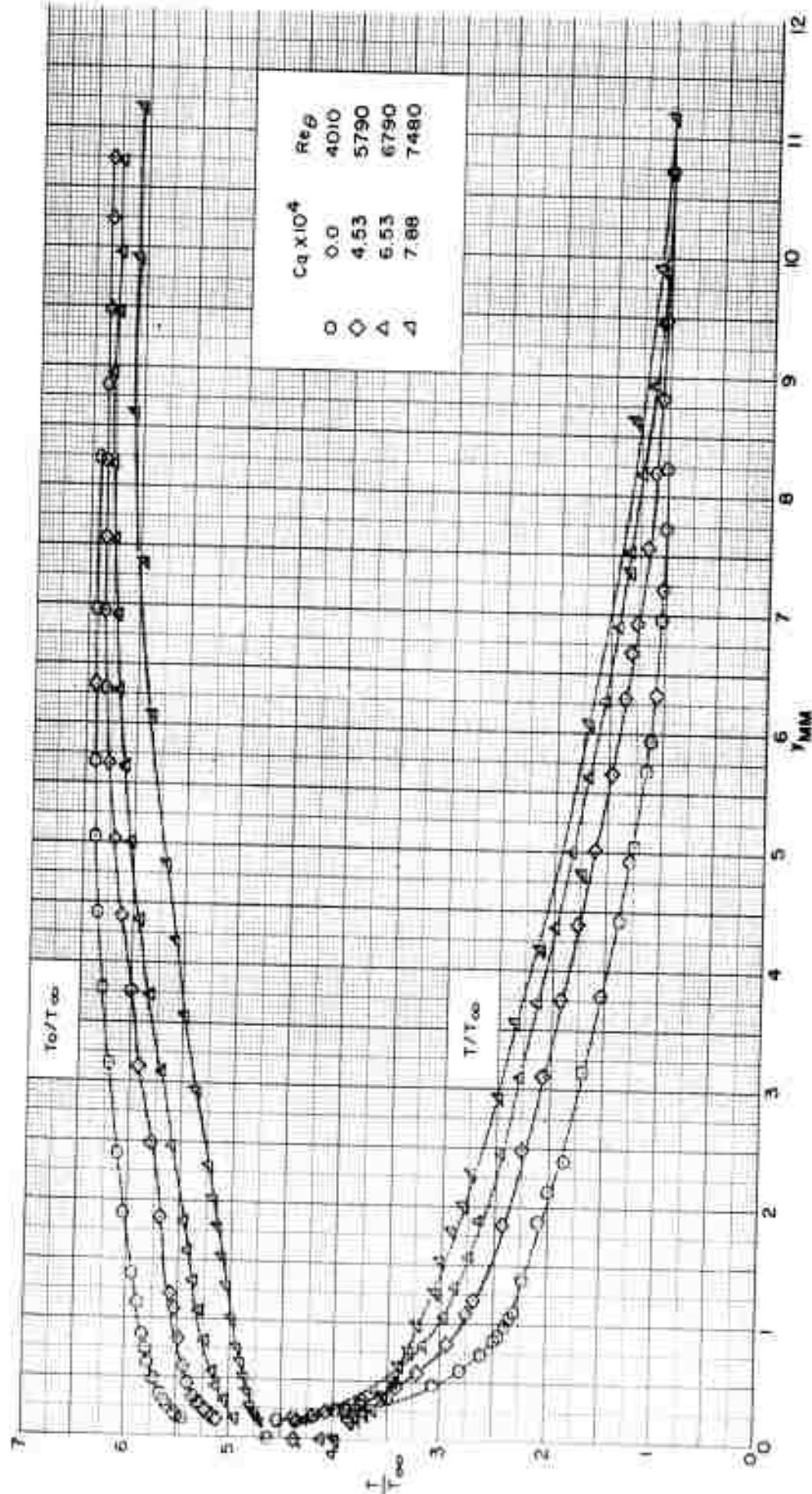


FIG. II TURBULENT BOUNDARY-LAYER TOTAL AND STATIC TEMPRATURE PROFILES WITH  
VARIOUS RATES OF AIR INJECTION  
 $Re_x = 5.4 \times 10^6$   $T_w/T_{\infty} = 4.23$

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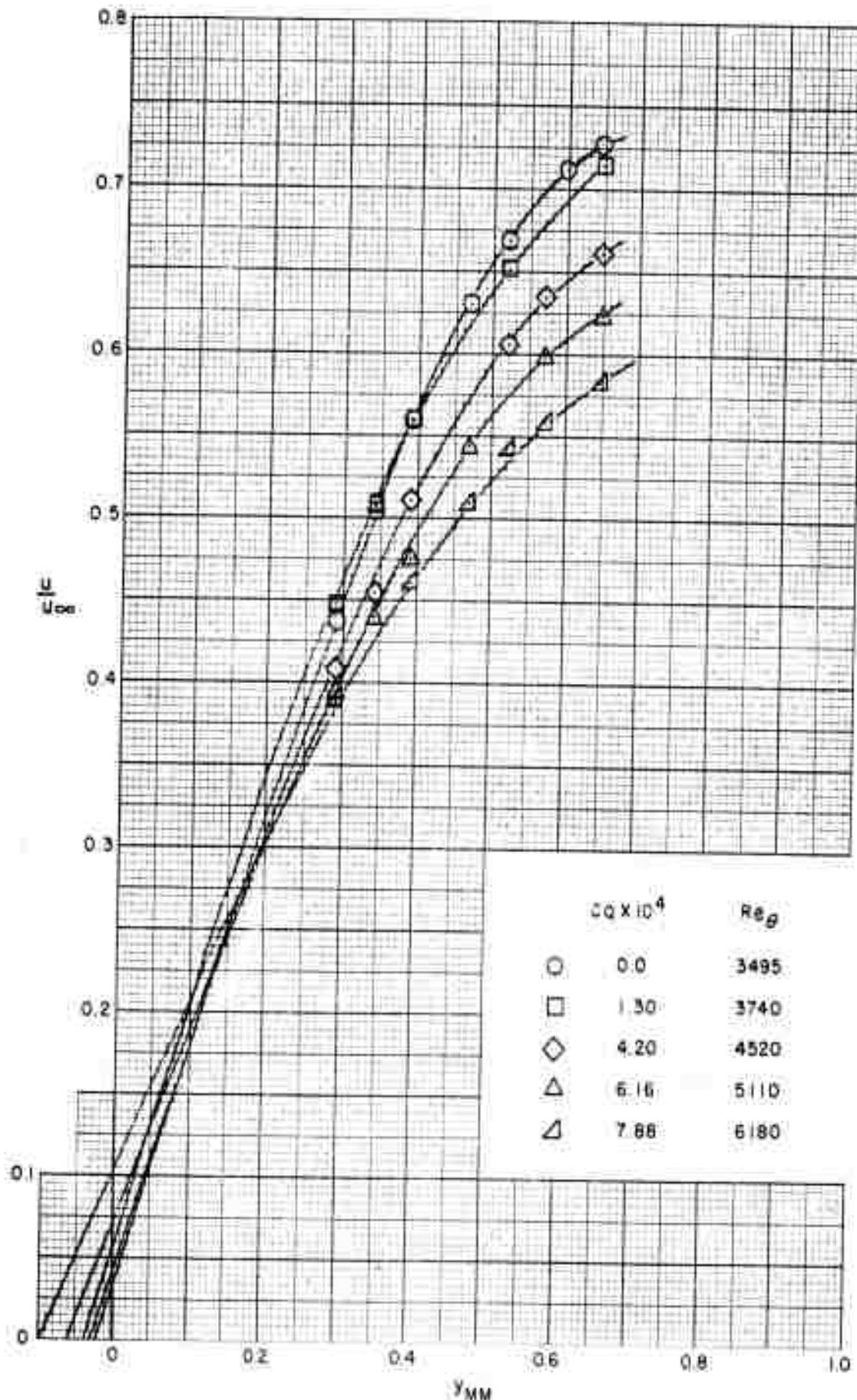


FIG. 12 DETAIL OF THE VELOCITY PROFILE,  $y < 1.0$  MM  
 $Re_x = 4.3 \times 10^6$ ,  $T_w/T_{\infty} = 5.04$

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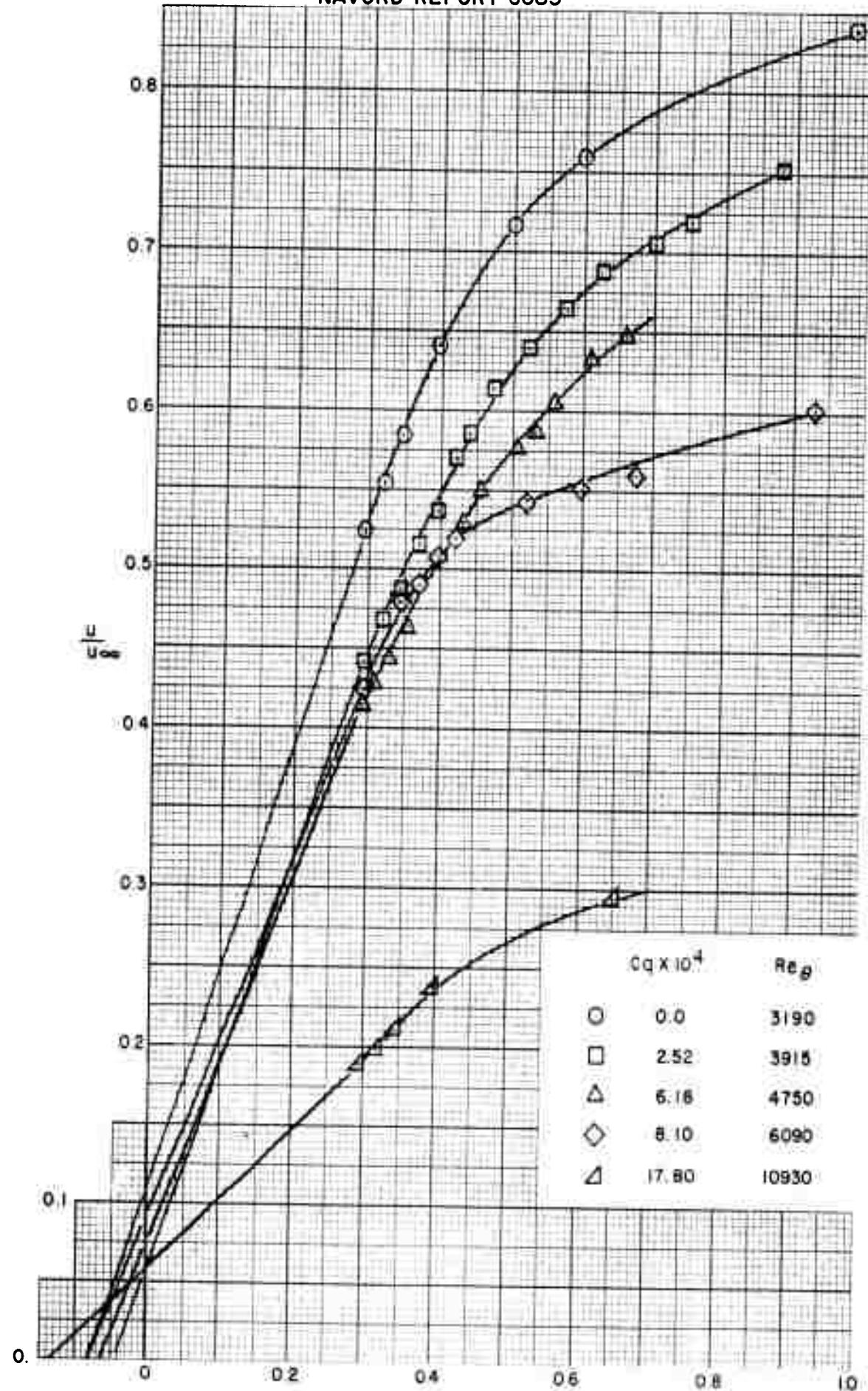


FIG. 13 DETAIL OF THE VELOCITY PROFILE,  $y < 1.0$  MM

$$Re_x = 4.2 \times 10^6, Tw/T_\infty = 4.23$$

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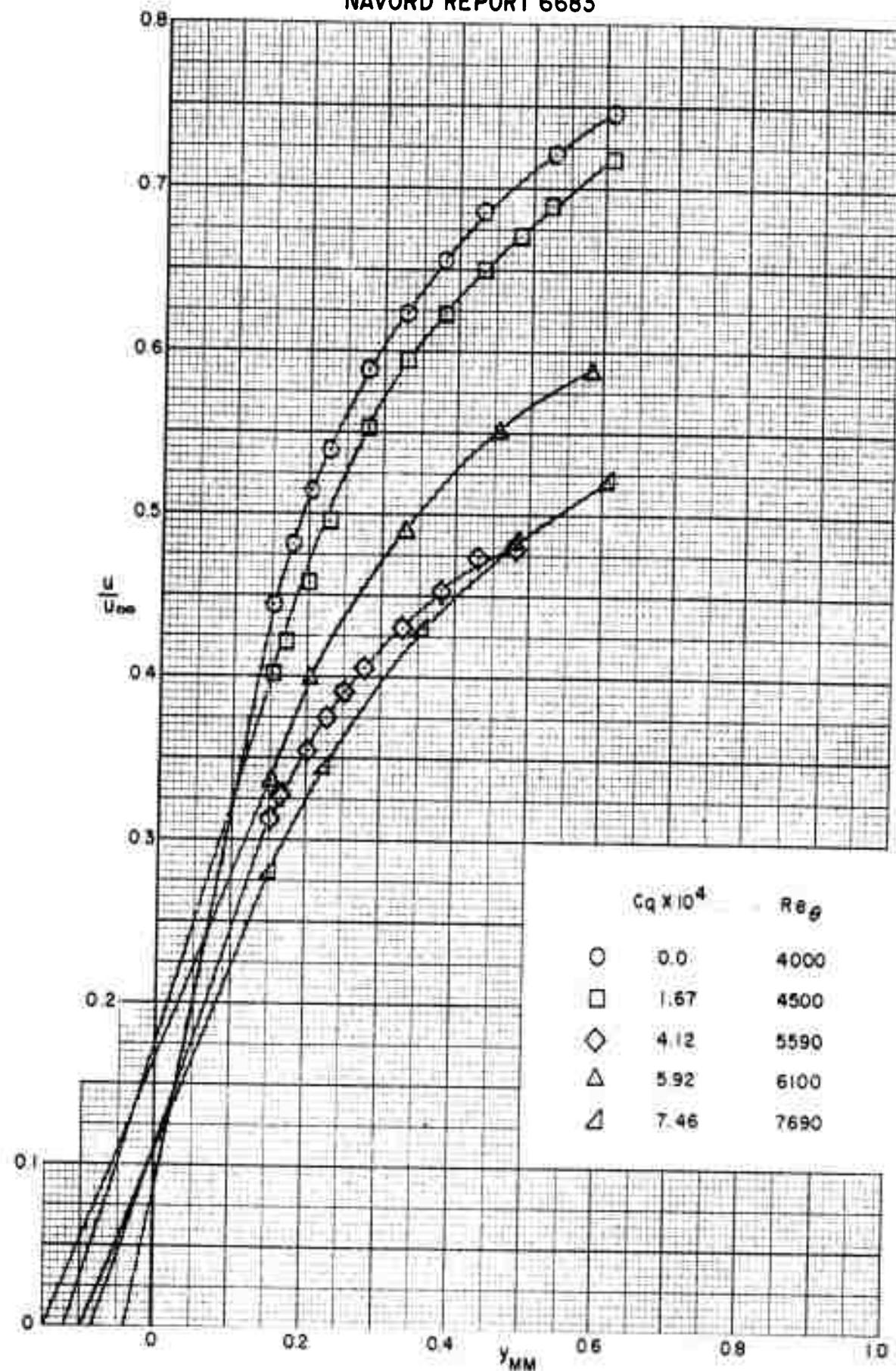


FIG. 14 DETAIL OF THE VELOCITY PROFILE,  $y < 1.0$  MM  
 $Re_x = 5.5 \times 10^6$ ,  $T_w/T_\infty = 5.04$

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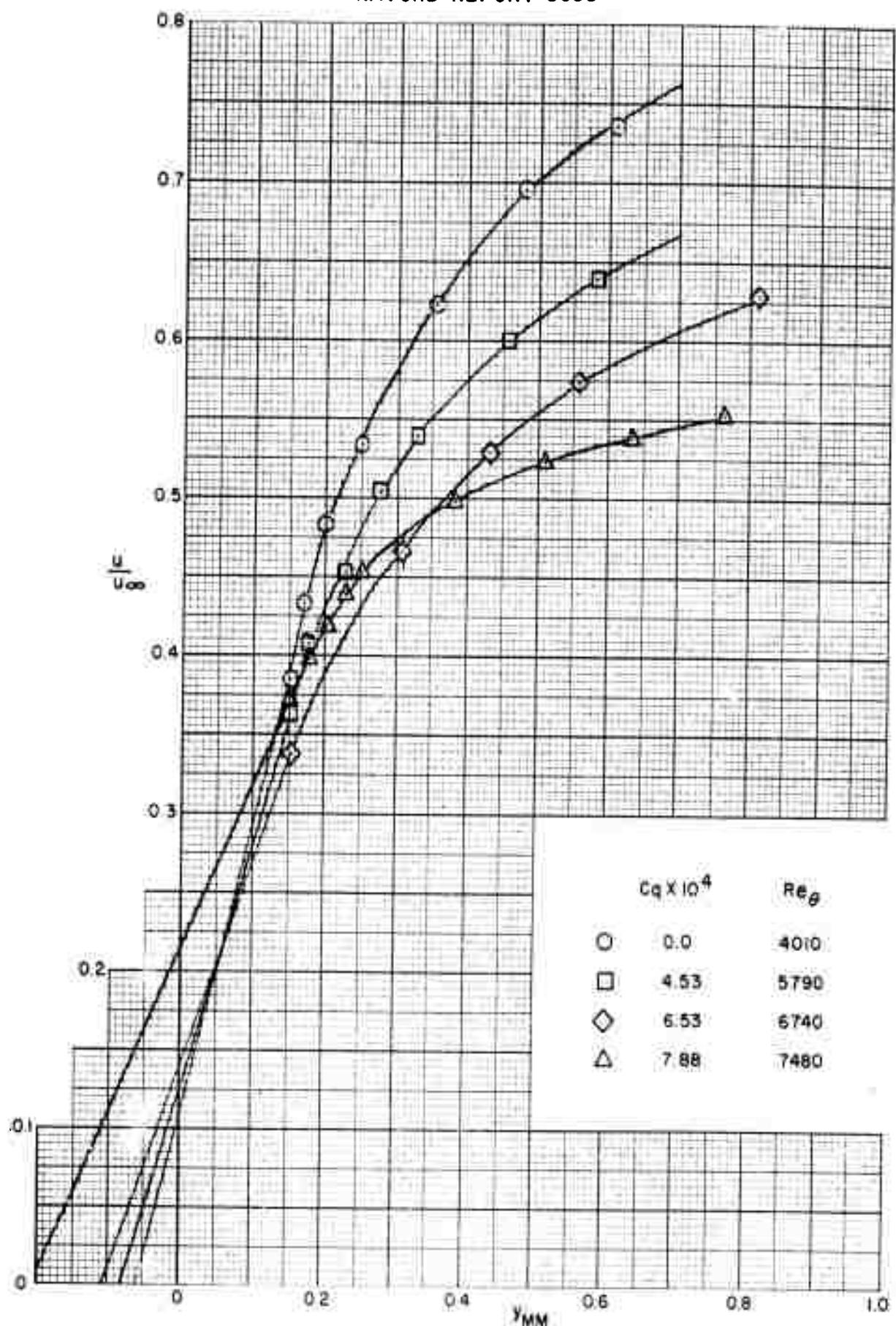


FIG.15 DETAIL OF THE VELOCITY PROFILE,  $y < 1.0$  MM  
 $Re_x = 5.4 \times 10^6$ ,  $T_w/T_{\infty} = 4.23$

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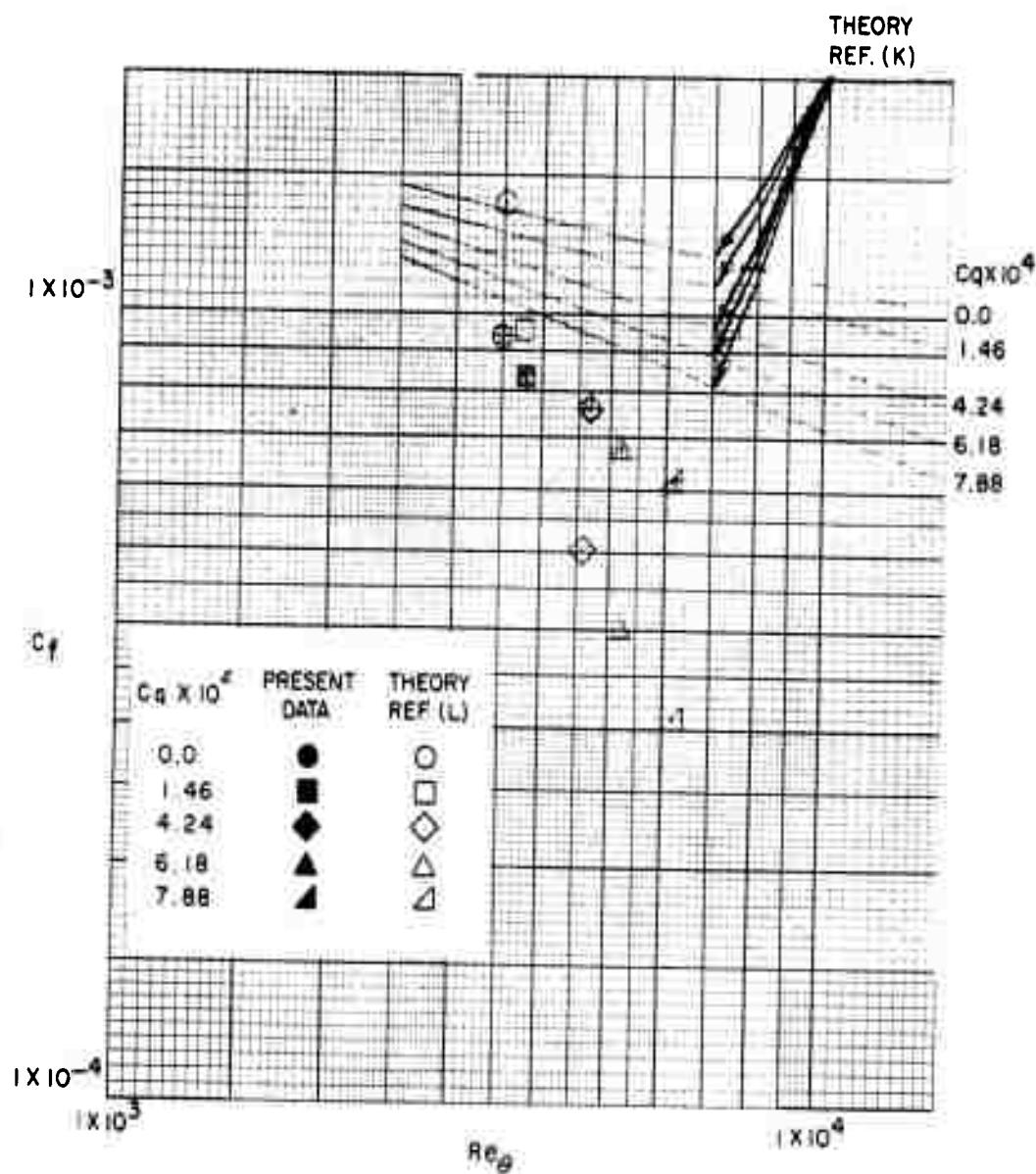


FIG. 16a SKIN-FRICTION COEFFICIENT VERSUS  
MOMENTUM THICKNESS REYNOLDS NUMBER  
 $Re_x = 4.3 \times 10^6 \quad Tw/T_\infty = 5.04$

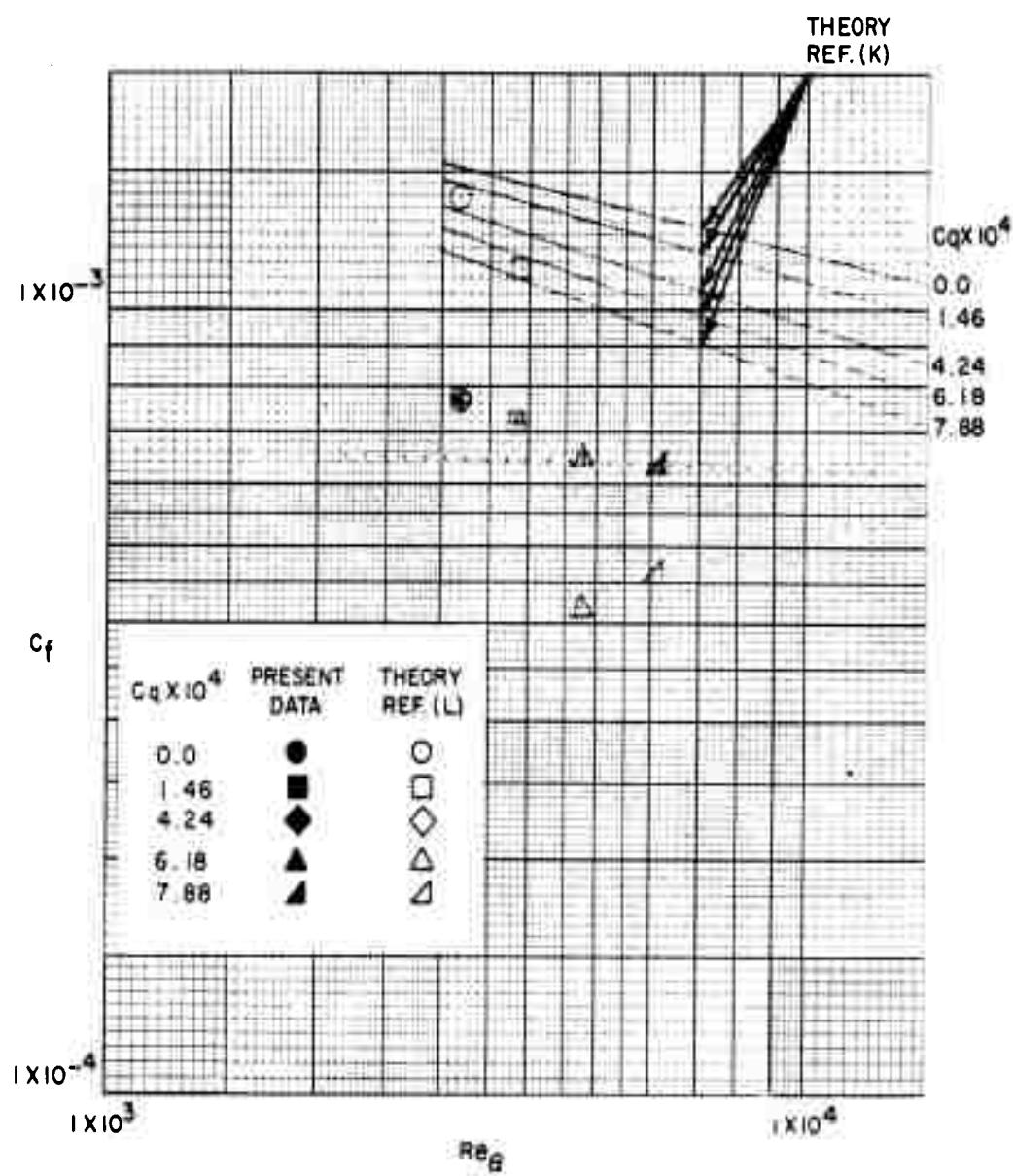


FIG. 16b SKIN-FRiction COEFFICIENT VERSUS  
MOMENTUM THICKNESS REYNOLDS NUMBER

$$Re_x = 4.2 \times 10^6, Tw/T_\infty = 4.23$$

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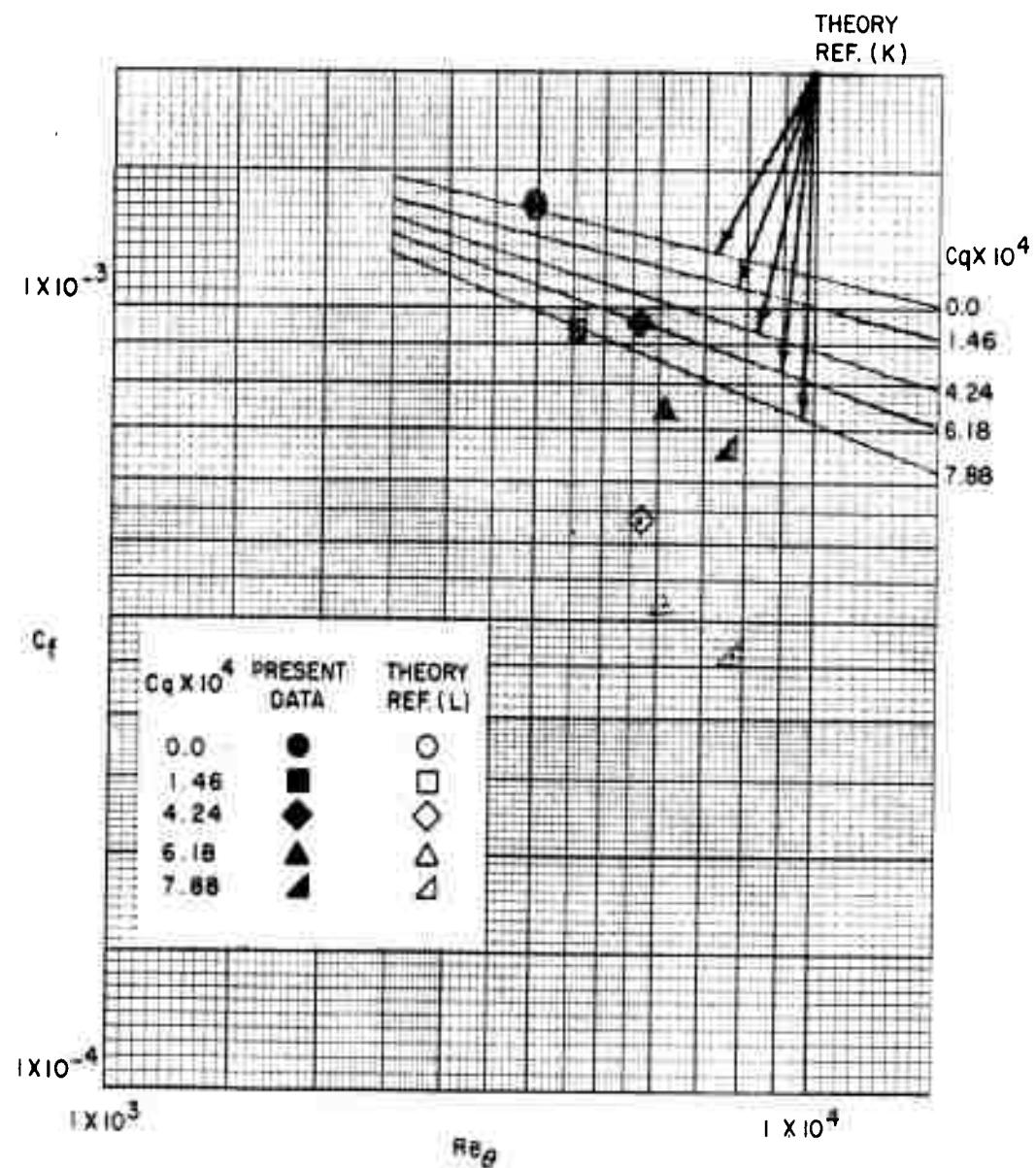


FIG. 17a SKIN-FRiction COEFFICIENT VERSUS  
MOMENTUM THICKNESS REYNOLDS NUMBER  
 $Re_x = 5.5 \times 10^6$ ,  $T_w/T_\infty = 5.04$

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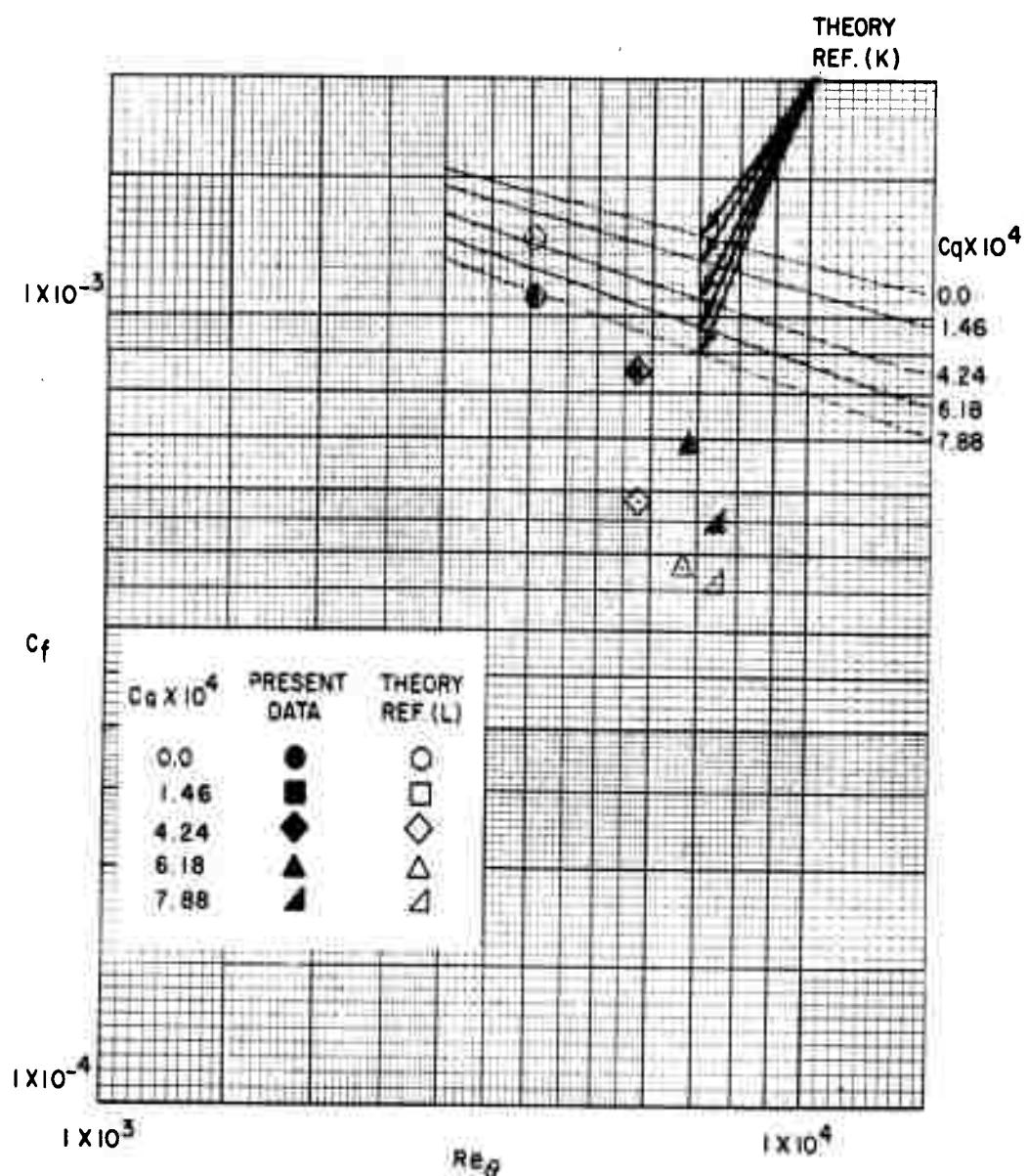


FIG. 17b SKIN-FRICTION COEFFICIENT VERSUS  
MOMENTUM THICKNESS REYNOLDS NUMBER

$$Re_x = 5.4 \times 10^6, Tw/T\infty = 4.23$$

## EXPERIMENTAL DATA

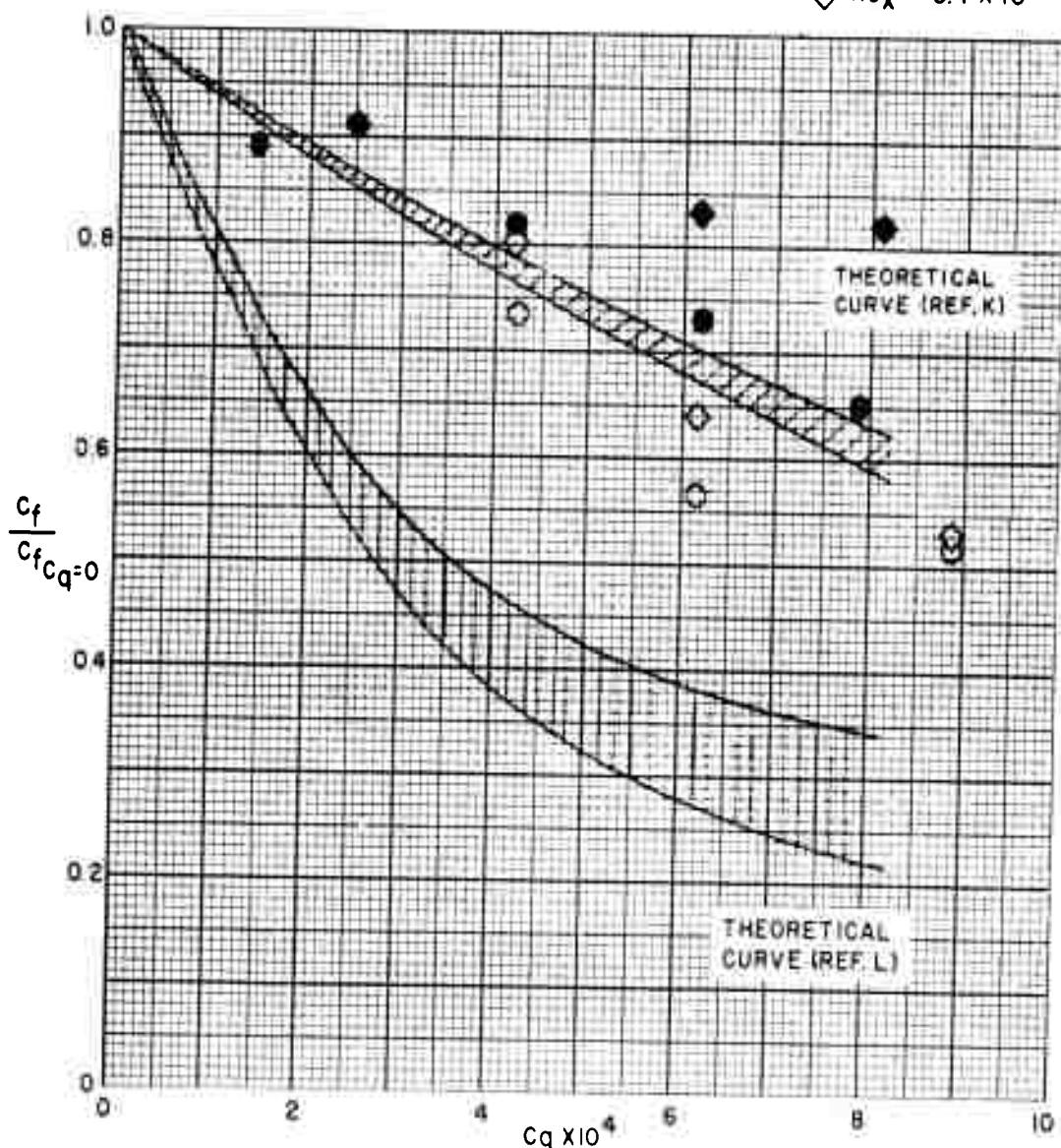
 $M_\infty = 5.1 \quad T_w/T_\infty = 5.04$  $\bullet \quad Re_x = 4.3 \times 10^6$  $\circ \quad Re_x = 5.5 \times 10^6$  $T_w/T_\infty = 4.23$  $\blacklozenge \quad Re_x = 4.2 \times 10^6$  $\diamond \quad Re_x = 5.4 \times 10^6$ 

FIG.18 VARIATION OF SKIN-FRICTION RATIO WITH AIR INJECTION RATE  
 $(C_f/C_{f0} = \text{SKIN-FRICTION COEFFICIENT FOR ZERO INJECTION RATE})$

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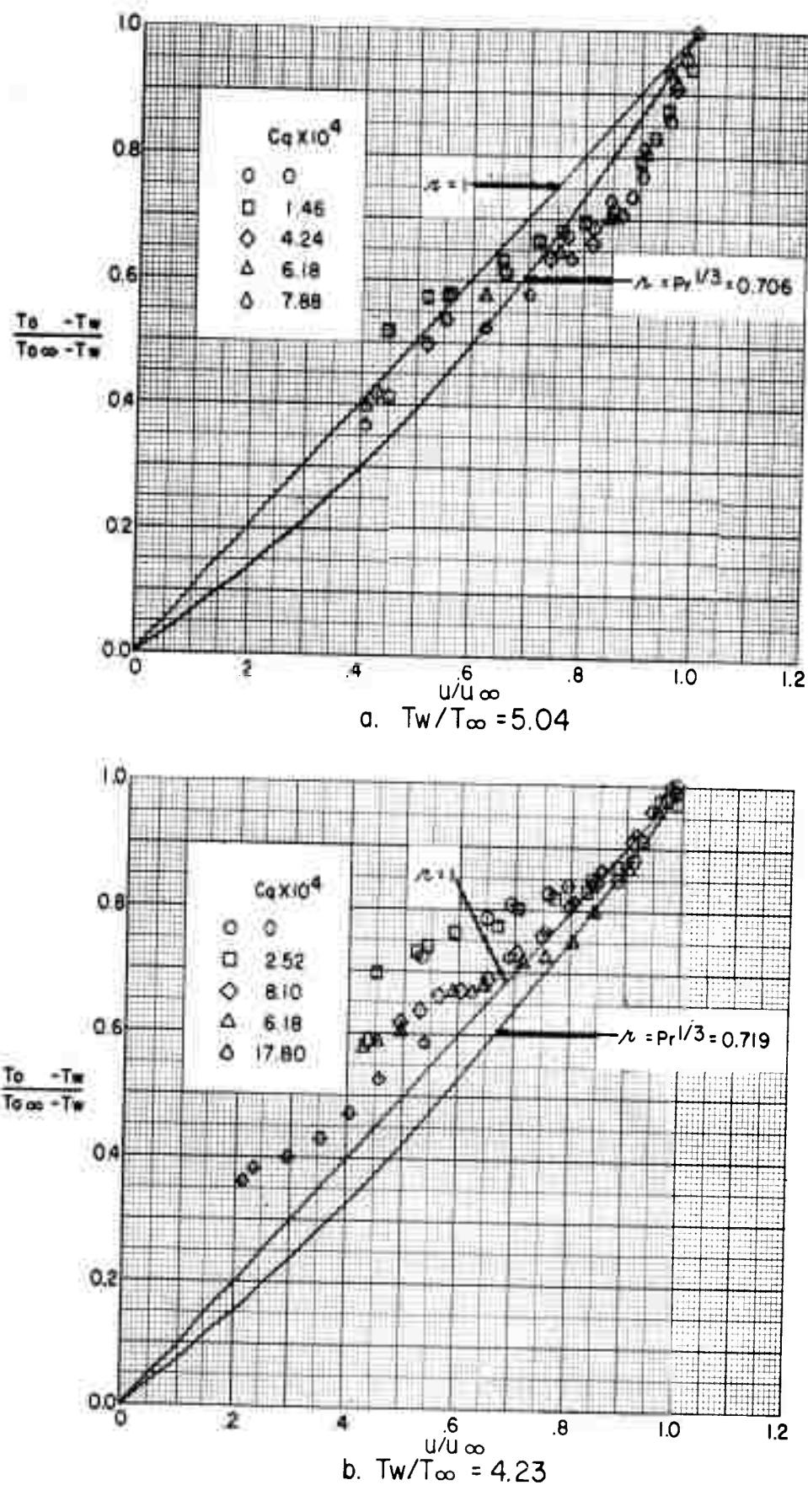
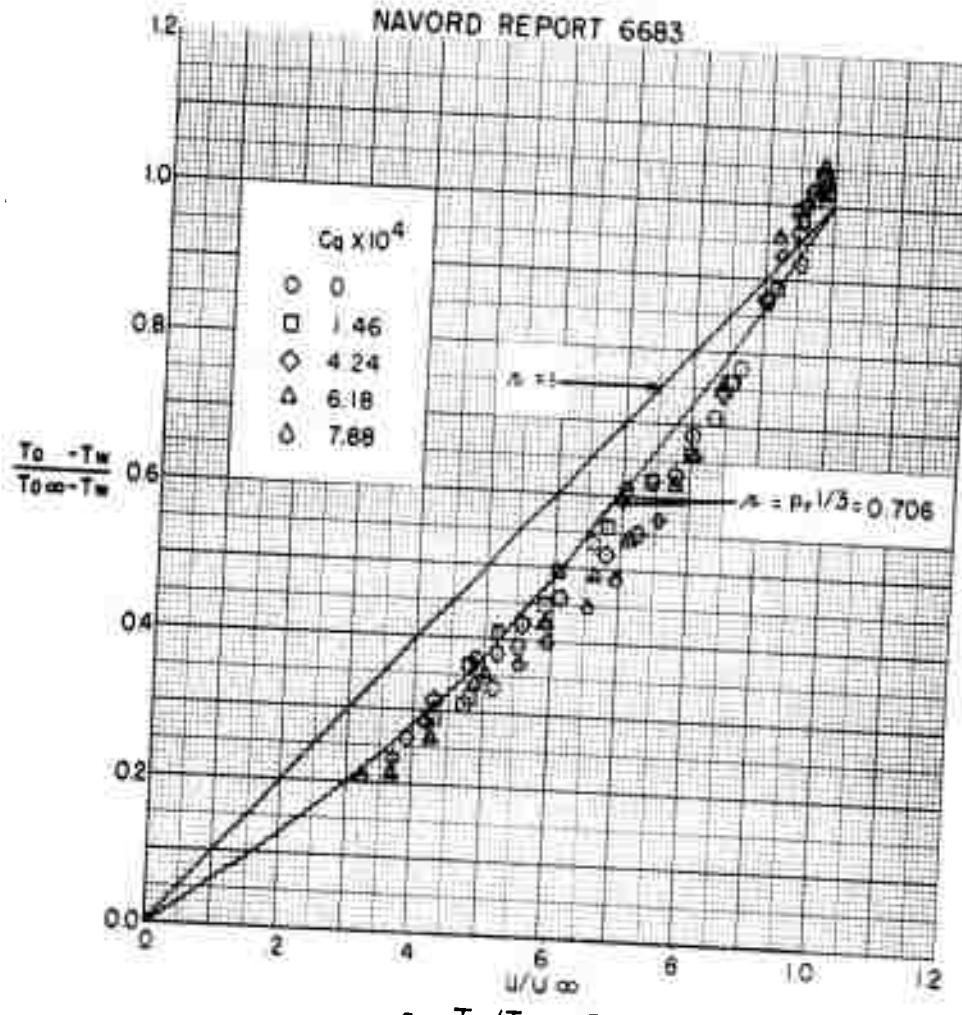
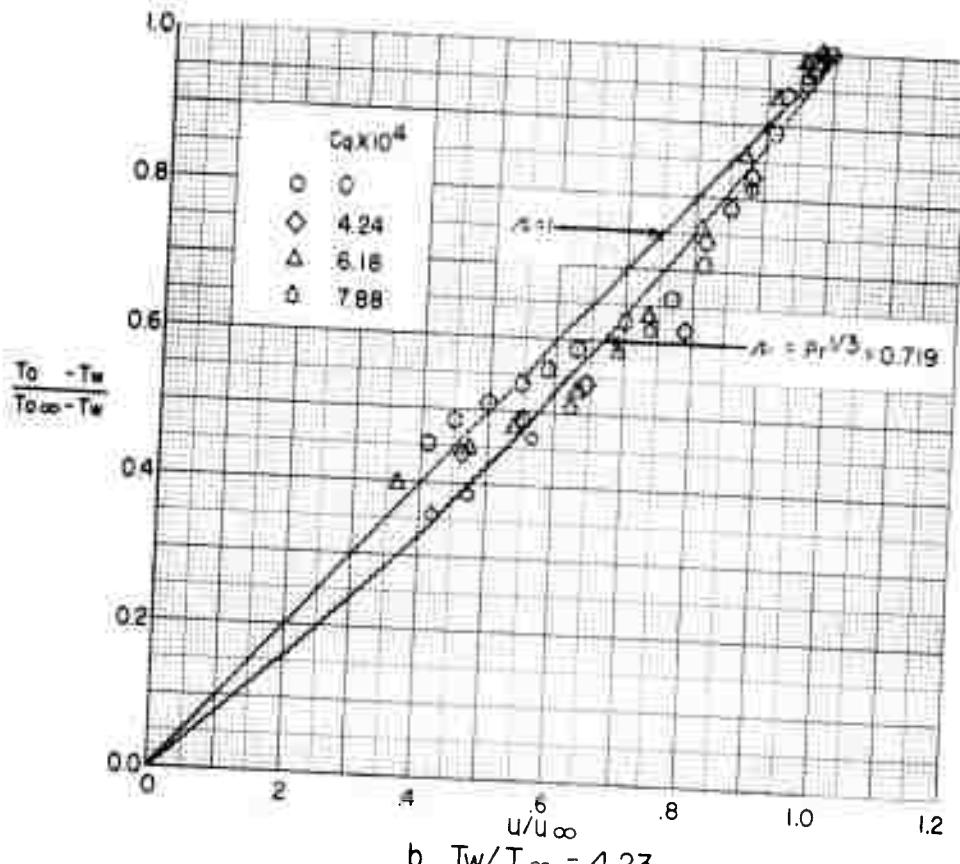


FIG. 19 COMPARISON OF EXPERIMENTAL AND THEORETICAL  
TEMPERATURE-VELOCITY DISTRIBUTIONS  
 $M_{\infty} = 5.08$   $Re_x = 4.2 \times 10^6$

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a.  $T_w/T_\infty = 5.04$



b.  $T_w/T_\infty = 4.23$

FIG. 20 COMPARISON OF EXPERIMENTAL AND THEORETICAL  
TEMPERATURE-VELOCITY DISTRIBUTIONS  
 $M_\infty = 5.08$   $Re_x = 5.4 \times 10^6$

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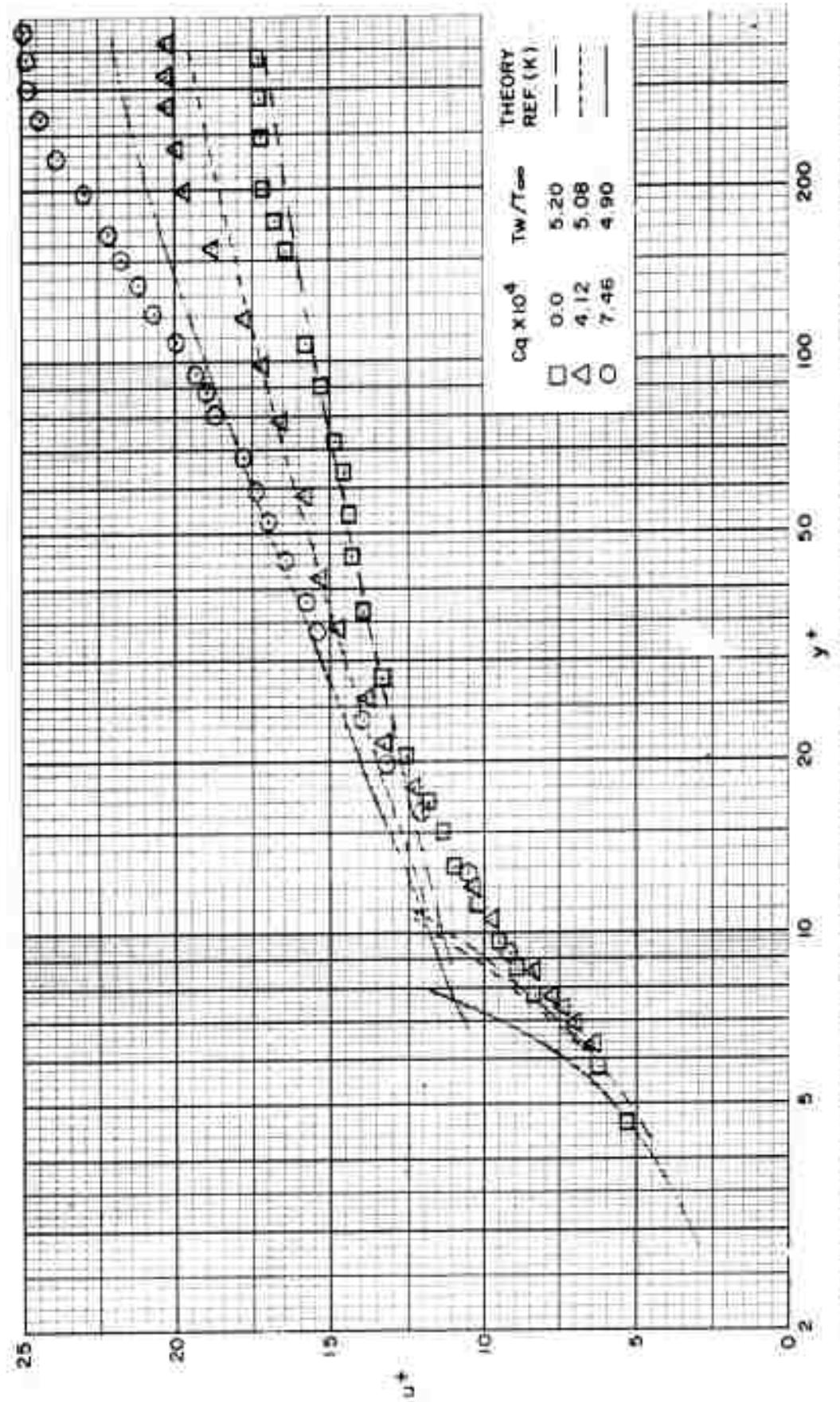


FIG. 21 COMPARISON OF EXPERIMENTAL AND THEORETICAL VELOCITY PROFILES

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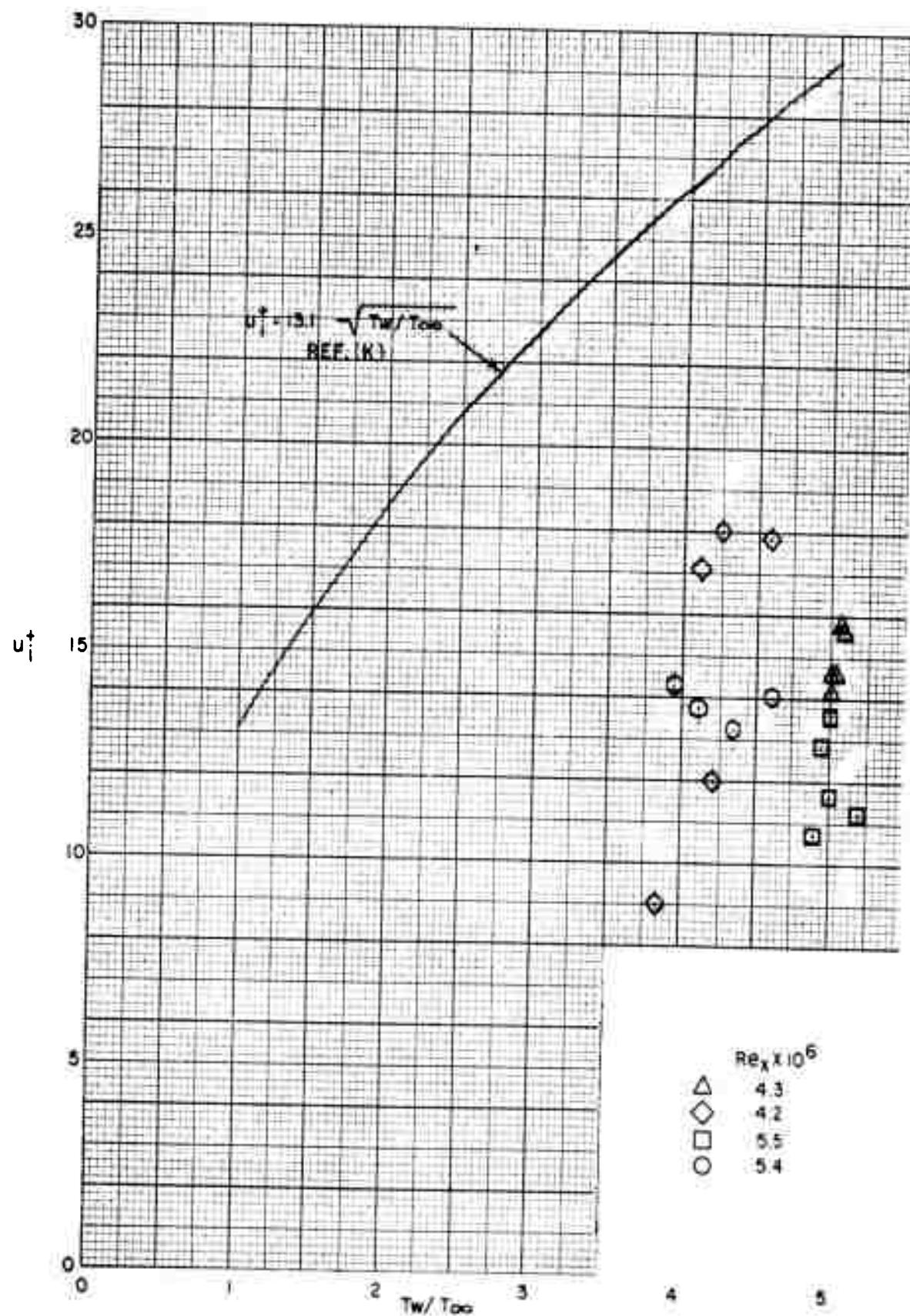


FIG. 22 INTERFACE NON-DIMENSIONAL VELOCITY,  $u_i^+$ , VERSUS  $T_w/T_\infty$

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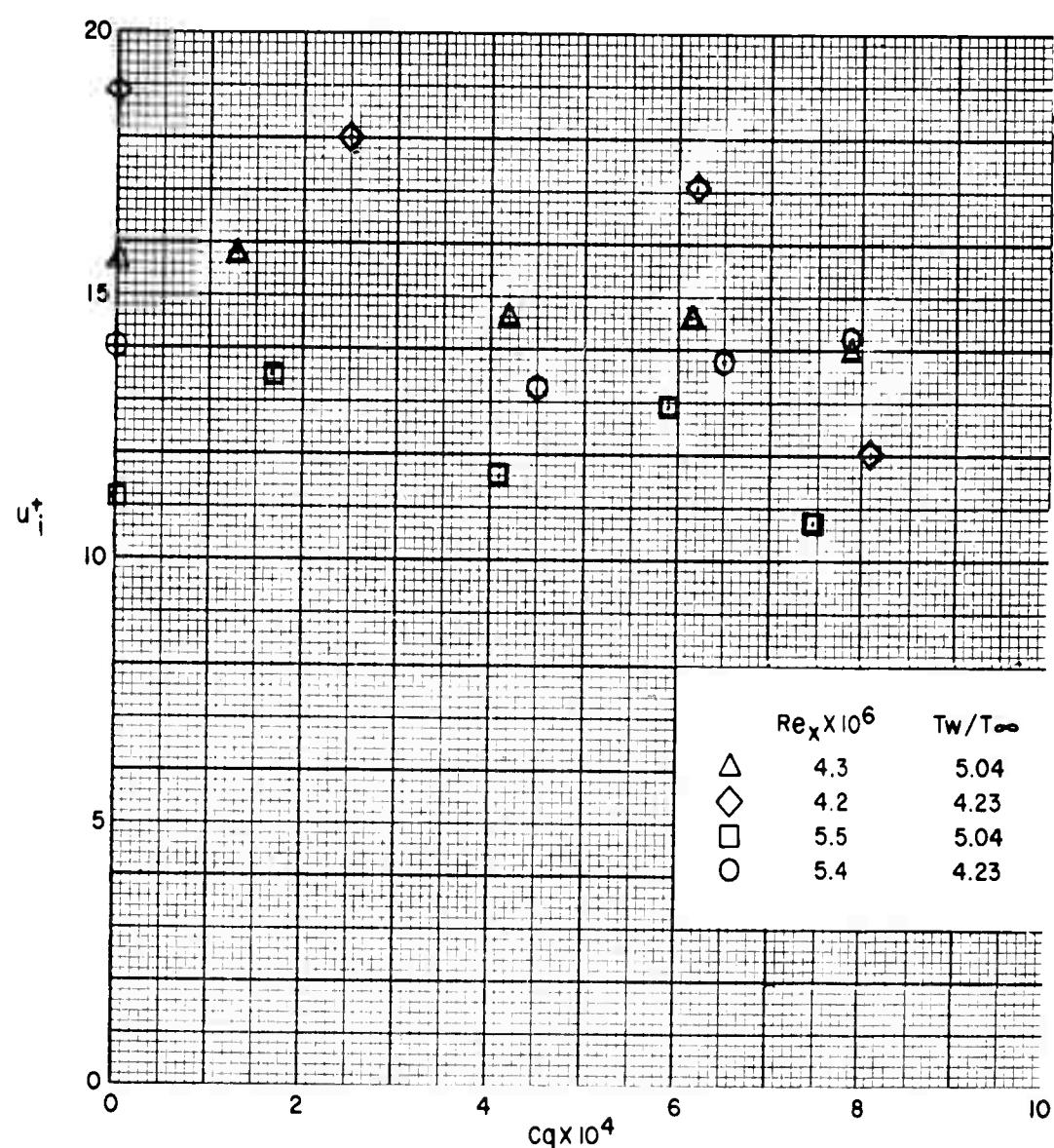


FIG. 23 INTERFACE NON-DIMENSIONAL VELOCITY,  $u_i^+$ , VERSUS INJECTION RATE  $C_q$

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TABLE I

$M_{\infty} = 5.05$	$P_0 = 7.40$	atm	
$Re_\theta = 3506$	$T_0 = 376.5$	°K	
$C_q = 0$	$T_{\infty} = 61.71$	°K	
$(T_e - T_w) / T_e = 0.0847$	$U_{\infty} = 795.4$	m/sec	
$T_w / T_{\infty} = 5.085$			
Y (mm)	M	T/T <sub>∞</sub>	U/U <sub>∞</sub>
12.71	5.05	1.000	1.000
11.44	5.04	1.004	1.000
10.17	5.03	1.007	.999
8.903	5.03	1.007	.999
8.268	5.02	1.011	.999
7.633	5.02	1.011	.999
6.998	5.00	1.018	.999
6.363	4.97	1.026	.997
6.109	4.95	1.032	.996
5.728	4.90	1.047	.993
5.093	4.75	1.097	.985
4.458	4.52	1.182	.973
3.823	4.21	1.313	.955
3.188	3.89	1.469	.934
3.061	3.82	1.507	.928
2.553	3.55	1.667	.908
2.299	3.41	1.758	.895
2.048	3.29	1.843	.884
1.791	3.20	1.907	.875
1.537	3.10	1.985	.865
1.410	3.04	2.035	.859
1.283	2.94	2.122	.848
1.156	2.82	2.233	.834
1.029	2.68	2.373	.818
.902	2.52	2.545	.796
.775	2.34	2.753	.769
.648	2.13	3.014	.732
.597	2.05	3.122	.717
.521	1.80	3.462	.663
.470	1.69	3.617	.636
.394	1.44	3.986	.568
.343	1.26	4.231	.512
.292	1.07	4.476	.449

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TABLE 2

$M_{\infty} = 5.06$	$P_0 = 7.48$	atm	
$Re\theta = 3739$	$T_0 = 378.2$	$^{\circ}\text{K}$	
$C_q = 1.30 \times 10^{-4}$	$T_{\infty} = 61.56$	$^{\circ}\text{K}$	
$(T_e - T_w) / T_e = 0.0911$	$U_{\infty} = 797.6$	m/sec	
$T_w / T_{\infty} = 5.070$			
Y (mm)	M	T/T <sub>∞</sub>	U/U <sub>∞</sub>
12.71	5.07	1.000	1.000
11.44	5.06	1.004	1.000
10.17	5.04	1.010	.999
9.538	5.03	1.014	.999
8.903	5.02	1.018	.999
8.268	5.03	1.014	.999
7.633	5.02	1.018	.999
6.998	5.00	1.024	.999
6.363	4.95	1.041	.998
5.728	4.83	1.080	.996
5.093	4.63	1.152	.980
4.458	4.37	1.255	.966
4.077	4.20	1.330	.955
3.823	4.06	1.398	.947
3.188	3.75	1.563	.925
2.553	3.42	1.770	.897
2.299	3.28	1.869	.885
2.045	3.19	1.935	.875
1.791	3.08	2.020	.863
1.537	2.97	2.109	.851
1.283	2.80	2.270	.831
1.156	2.69	2.376	.818
1.029	2.56	2.515	.801
.902	2.42	2.675	.781
.775	2.27	2.855	.757
.648	2.04	3.155	.715
.521	1.75	3.564	.652
.394	1.42	4.053	.564
.343	1.26	4.316	.516
.292	1.05	4.609	.445

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TABLE 3

$M_\infty = 5.08$	$P_0 = 7.67 \text{ atm}$		
$Re_\theta = 4523$	$T_0 = 377.5^\circ\text{K}$		
$C_q = 4.20 \times 10^{-4}$	$T_\infty = 61.27^\circ\text{K}$		
$(T_e - T_w)/T_e = 0.0984$	$U_\infty = 797.2 \text{ m/sec}$		
	$T_w/T_\infty = 5.043$		
$Y$ (mm)	$M$	$T/T_\infty$	$U/U_\infty$
12.713	5.080	1.000	1.000
11.443	5.061	1.007	1.000
10.808	5.056	1.009	1.000
10.173	5.045	1.012	.999
9.538	5.033	1.017	.999
8.903	5.029	1.018	.999
8.268	5.015	1.022	.998
7.633	4.972	1.036	.996
6.998	4.911	1.056	.994
6.363	4.791	1.094	.987
5.728	4.576	1.173	.976
5.093	4.322	1.278	.962
4.458	4.028	1.418	.944
4.077	3.847	1.513	.932
3.823	3.738	1.574	.923
3.188	3.434	1.764	.898
2.553	3.132	1.979	.867
2.299	3.028	2.060	.856
2.045	2.932	2.140	.844
1.791	2.800	2.260	.829
1.537	2.669	2.388	.811
1.410	2.603	2.455	.803
1.283	2.503	2.563	.789
1.156	2.388	2.698	.772
1.029	2.295	2.809	.757
.902	2.168	2.966	.735
.775	2.017	3.166	.706
.648	1.829	3.425	.666
.521	1.611	3.737	.613
.394	1.289	4.204	.520
.343	1.122	4.434	.465
.292	1.005	4.586	.424

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TABLE 4

$M_\infty = 5.09$	$P_0 = 7.795$	atm	
$Re_\theta = 5108$	$T_0 = 379.9$	$^{\circ}\text{K}$	
$C_q = 6.16 \times 10^{-4}$	$T_{\infty} = 61.47$	$^{\circ}\text{K}$	
$(T_e - T_w) / T_e = 0.1064$	$U_\infty = 800.12$	m/sec	
$T_w / T_{\infty} = 5.013$			
$Y$ (mm)	$M$	$T / T_{\infty}$	$U / U_\infty$
12.71	5.09	1.000	1.000
11.44	5.06	1.010	.999
10.17	5.04	1.017	.999
9.538	5.03	1.020	.998
8.903	5.02	1.023	.998
8.268	4.96	1.043	.995
7.633	4.89	1.066	.992
6.998	4.78	1.103	.986
6.363	4.59	1.173	.977
5.728	4.32	1.285	.962
5.347	4.15	1.363	.952
5.093	4.05	1.410	.945
4.458	3.76	1.566	.924
3.823	3.48	1.735	.901
3.188	3.20	1.928	.874
3.061	3.16	1.966	.869
2.553	2.94	2.137	.844
2.299	2.82	2.249	.829
2.048	2.71	2.350	.816
1.791	2.58	2.482	.799
1.537	2.44	2.637	.778
1.283	2.28	2.827	.753
1.156	2.18	2.953	.736
1.029	2.06	3.110	.714
.902	1.96	3.246	.694
.775	1.82	3.440	.663
.648	1.67	3.646	.628
.521	1.46	3.947	.571
.470	1.39	4.051	.550
.394	1.18	4.341	.484
.343	1.08	4.471	.450
.292	.972	4.611	.410

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TABLE 5

$M_\infty = 5.09$	$P_0 = 7.87$	atm	
$Re_\theta = 6178$	$T_0 = 377.46$	$^{\circ}\text{K}$	
$C_q = 7.89 \times 10^{-4}$	$T_\infty = 61.07$	$^{\circ}\text{K}$	
$(T_e - T_w)/T_e = 0.1065$	$U_\infty = 797.6$	m/sec	
	$T_w/T_\infty = 5.027$		
Y (mm)	M	T/T <sub>∞</sub>	U/U <sub>∞</sub>
13.983	5.09	1.000	1.000
12.713	5.07	1.006	.999
11.443	5.05	1.013	.998
10.808	5.02	1.023	.998
10.173	5.01	1.026	.997
9.538	5.00	1.029	.997
8.903	4.94	1.048	.994
8.268	4.85	1.079	.990
7.633	4.74	1.117	.984
6.998	4.55	1.188	.974
6.617	4.42	1.243	.968
6.363	4.31	1.291	.962
5.728	4.01	1.431	.947
5.093	3.76	1.578	.926
4.458	3.52	1.718	.905
3.823	3.24	1.906	.879
3.188	2.97	2.120	.850
2.807	2.77	2.299	.825
2.553	2.69	2.374	.814
2.299	2.57	2.405	.798
2.045	2.43	2.644	.776
1.791	2.32	2.763	.758
1.537	2.18	2.934	.734
1.283	2.03	3.123	.705
1.156	1.96	3.215	.690
1.036	1.89	3.310	.676
1.029	1.87	3.341	.672
.902	1.79	3.451	.653
.826	1.72	3.547	.636
.775	1.66	3.633	.622
.648	1.54	3.807	.590
.572	1.45	3.940	.565
.521	1.40	4.014	.551
.470	1.29	4.166	.517
.394	1.15	4.363	.472
.292	.93	4.640	.394

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TABLE 6

$M_\infty = 5.09$	$P_0 = 7.50$	atm	
$Re_\theta = 3186$	$T_0 = 369.3$	°K	
$C_q = 0$	$T_\infty = 59.83$	°K	
$(T_e - T_w)/T_e = 0.183$	$U_\infty = 788.8$	m/sec	
$T_w/T_\infty = 4.595$			
$Y$ (mm)	$M$	$T/T_\infty$	$U/U_\infty$
8.727	5.086	1.000	1.000
6.187	5.051	1.012	.999
5.552	4.968	1.038	.995
4.917	4.796	1.095	.987
4.282	4.528	1.196	.974
3.647	4.245	1.318	.958
3.266	4.057	1.407	.946
3.012	3.928	1.474	.938
2.758	3.796	1.547	.928
2.504	3.651	1.632	.917
2.377	3.561	1.695	.912
2.250	3.530	1.713	.908
2.123	3.415	1.787	.898
1.996	3.376	1.813	.894
1.869	3.331	1.845	.890
1.742	3.302	1.866	.887
1.615	3.277	1.885	.885
1.488	3.265	1.894	.883
1.361	3.204	1.944	.878
1.234	3.143	1.995	.873
1.107	2.946	2.170	.853
.980	2.787	2.323	.835
.853	2.679	2.436	.822
.726	2.478	2.658	.794
.599	2.233	2.957	.755
.472	1.922	3.377	.694
.396	1.720	3.667	.648
.345	1.512	3.974	.593
.320	1.320	4.257	.535
.295	1.310	4.267	.532

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TABLE 7

$M_\infty = 5.05$	$P_0 = 7.26$	atm	
$R\theta_\theta = 3915$	$T_0 = 372.62$	°K	
$C_q = 2.52 \times 10^{-4}$	$T_\infty = 61.07$	°K	
$(T_e - T_w)/T_e = 0.236$	$U_\infty = 791.29$	m/sec	
$T_w/T_\infty = 4.257$			
$Y$ (mm)	$M$	$T/T_\infty$	$U/U_\infty$
7.991	5.049	1.000	1.000
7.356	5.027	1.008	1.000
6.721	4.955	1.033	.997
6.086	4.866	1.084	.991
5.451	4.572	1.170	.979
4.816	4.300	1.281	.964
4.181	4.003	1.422	.945
3.927	3.891	1.481	.938
3.673	3.779	1.542	.929
3.419	3.652	1.616	.919
3.165	3.525	1.695	.909
2.911	3.354	1.812	.894
2.657	3.260	1.879	.885
2.403	3.147	1.964	.873
2.149	3.030	2.056	.860
1.895	2.945	2.126	.850
1.641	2.850	2.208	.839
1.463	2.690	2.364	.819
1.387	2.716	2.388	.822
1.260	2.640	2.414	.812
1.133	2.534	2.528	.798
1.006	2.428	2.647	.782
.955	2.315	2.782	.765
.879	2.310	2.786	.764
.752	2.082	3.076	.723
.701	2.016	3.165	.710
.625	2.000	3.186	.707
.574	1.840	3.406	.672
.523	1.802	3.460	.664
.473	1.658	3.666	.629
.396	1.344	4.123	.540
.345	1.204	4.322	.496
.320	1.147	4.401	.476
.295	1.072	4.504	.450

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TABLE 8

$M_\infty$	4.94	$P_0$	6.64	atm
$Re_\theta$	4747	$T_0$	375.32	°K
$C_q$	$6.18 \times 10^{-4}$	$T_\infty$	63.92	°K
$(T_e - T_w)/T_e$	0.235	$U_\infty$	791.39	m/sec
		$T_w/T_\infty$	4.107	
$Y$ (mm)	$M$	$T/T_\infty$	$U/U_\infty$	
9.423	4.937	1.000	1.000	
9.042	4.927	1.003	1.000	
8.788	4.910	1.008	.999	
8.534	4.910	1.008	.999	
8.153	4.862	1.024	.997	
7.772	4.838	1.032	.995	
7.518	4.787	1.050	.994	
6.883	4.654	1.096	.987	
6.248	4.430	1.182	.976	
5.613	4.196	1.280	.962	
4.978	3.942	1.399	.944	
4.343	3.707	1.521	.926	
3.708	3.440	1.676	.902	
3.073	3.199	1.830	.877	
2.438	2.944	2.018	.847	
2.184	2.848	2.093	.835	
1.930	2.759	2.165	.822	
1.676	2.640	2.269	.806	
1.549	2.566	2.340	.795	
1.422	2.484	2.419	.783	
1.295	2.392	2.514	.768	
1.168	2.284	2.635	.751	
1.041	2.185	2.751	.734	
.914	2.070	2.891	.713	
.864	2.017	2.956	.702	
.787	1.935	3.060	.686	
.711	1.840	3.182	.665	
.660	1.768	3.275	.648	
.610	1.721	3.338	.637	
.533	1.547	3.572	.592	
.406	1.222	4.003	.495	
.330	1.090	4.151	.453	
.292	1.027	4.229	.428	

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TABLE 9

$M_\infty$	4.98	$P_0$	7.09	atm
$Re_\theta$	6092	$T_0$	370.55	°K
$C_q$	$8.10 \times 10^{-4}$	$T_\infty$	62.22	°K
$(T_e - T_w)/T_e$	0.225	$U_\infty$	787.28	m/sec
		$T_w/T_\infty$	4.219	
$Y$ (mm)	$M$	$T/T_\infty$	$U/U_\infty$	
10.84	4.978	1.000	1.000	
10.20	4.971	1.002	1.000	
9.566	4.952	1.009	.999	
9.058	4.876	1.035	.997	
8.804	4.846	1.046	.996	
8.296	4.717	1.092	.990	
7.788	4.657	1.113	.987	
7.534	4.531	1.158	.980	
7.280	4.356	1.232	.971	
7.026	4.230	1.288	.964	
6.772	4.163	1.317	.960	
6.264	3.951	1.420	.946	
5.756	3.716	1.548	.929	
5.248	3.483	1.690	.910	
4.740	3.250	1.850	.888	
4.232	3.044	2.005	.866	
3.724	2.845	2.170	.842	
3.216	2.624	2.372	.812	
2.962	2.542	2.450	.799	
2.708	2.428	2.566	.781	
2.454	2.349	2.646	.768	
2.200	2.216	2.794	.744	
1.946	2.107	2.918	.723	
1.692	2.004	3.040	.702	
1.438	1.882	3.190	.675	
1.184	1.777	3.328	.651	
.930	1.600	3.568	.607	
.676	1.446	3.782	.565	
.599	1.410	3.830	.554	
.523	1.389	3.857	.548	
.422	1.325	3.942	.528	
.390	1.297	3.978	.520	
.371	1.222	4.081	.496	
.345	1.196	4.113	.487	
.295	1.047	4.301	.436	

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TABLE 10

$M_{\infty} = 4.85$	$P_0 = 7.07$	atm	
$Re_\theta = 10,931$	$T_0 = 375.32$	$^{\circ}\text{K}$	
$C_q = 17.8 \times 10^{-4}$	$T_\infty = 65.79$	$^{\circ}\text{K}$	
$(T_e - T_w)/T_e = 0.263$	$U_\infty = 788.73$	m/sec	
$T_w/T_\infty = 3.845$			
$Y$ (mm)	$M$	$T/T_\infty$	$U/U_\infty$
14.62	4.850	1.000	1.000
13.35	4.710	1.054	.997
12.84	4.626	1.084	.993
12.33	4.480	1.140	.986
11.83	4.268	1.227	.975
11.32	4.090	1.305	.963
10.81	3.815	1.442	.945
10.30	3.638	1.534	.929
9.794	3.510	1.601	.916
9.286	3.193	1.812	.886
8.753	2.926	2.011	.856
8.270	2.822	2.084	.840
7.762	2.590	2.288	.808
7.254	2.393	2.476	.776
6.746	2.270	2.605	.755
6.238	2.079	2.806	.718
5.730	1.966	2.927	.693
5.222	1.805	3.112	.657
4.714	1.682	3.250	.625
4.206	1.519	3.440	.581
3.444	1.366	3.597	.534
3.190	1.283	3.689	.508
2.682	1.124	3.856	.455
2.174	1.060	3.898	.432
1.666	.992	3.943	.406
1.158	.844	4.070	.351
.650	.707	4.176	.298
.396	.562	4.289	.240
.371	.552	4.297	.236
.345	.543	4.300	.232
.320	.514	4.319	.220
.295	.493	4.333	.212

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TABLE II

$M_{\infty} = 5.18$	$P_0 = 9.646$	atm	
$Re_\theta = 4000$	$T_0 = 379.0$	$^{\circ}\text{K}$	
$C_q = 0$	$T_\infty = 59.57$	$^{\circ}\text{K}$	
$(T_e - T_w)/T_e = 0.0995$	$U_\infty = 801.6$	m/sec	
	$T_w/T_\infty = 5.21$		
Y (mm)	M	T/T <sub>∞</sub>	U/U <sub>∞</sub>
9.373	5.18	1.000	1.000
8.738	5.18	1.000	1.000
8.103	5.17	1.004	1.000
7.468	5.13	1.018	.999
6.833	5.05	1.044	.996
6.198	4.89	1.103	.991
5.563	4.67	1.188	.983
4.928	4.40	1.306	.971
4.293	4.11	1.444	.954
3.658	3.82	1.599	.932
3.023	3.51	1.793	.907
2.515	3.27	1.953	.882
2.261	3.16	2.034	.870
2.007	3.04	2.127	.856
1.753	2.94	2.213	.844
1.499	2.85	2.294	.833
1.245	2.77	2.367	.823
.991	2.62	2.527	.804
.864	2.51	2.649	.789
.737	2.39	2.789	.771
.610	2.24	2.974	.746
.483	2.04	3.243	.709
.432	1.94	3.345	.685
.381	1.82	3.500	.659
.330	1.67	3.710	.623
.279	1.55	3.850	.589
.229	1.42	4.030	.553
.203	1.29	4.200	.513
.178	1.19	4.350	.480

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TABLE 12

$M_\infty = 5.14$	$P_0 = 8.05$	atm	
$Re_\theta = 4500$	$T_0 = 380.5$	°K	
$C_q = 1.69 \times 10^{-4}$	$T_\infty = 60.54$	°K	
$(T_e - T_w)/T_e = 0.1167$	$U_\infty = 801.9$	m/sec	
$T_w/T_\infty = 5.038$			
Y (mm)	M	T/T <sub>∞</sub>	U/U <sub>∞</sub>
10.64	5.14	1.000	1.000
9.373	5.14	1.000	1.000
8.738	5.12	1.007	1.000
8.103	5.09	1.018	.999
6.833	4.87	1.095	.992
6.198	4.68	1.169	.984
5.563	4.43	1.277	.974
4.928	4.15	1.408	.958
4.293	3.87	1.554	.939
3.658	3.58	1.725	.915
3.023	3.31	1.898	.887
2.388	3.02	2.119	.855
1.753	2.79	2.311	.825
1.499	2.70	2.393	.813
1.245	2.60	2.491	.798
.991	2.46	2.637	.777
.864	2.37	2.742	.763
.737	2.24	2.900	.743
.610	2.11	3.050	.717
.483	1.92	3.298	.678
.432	1.81	3.448	.654
.381	1.72	3.573	.632
.330	1.61	3.725	.604
.279	1.47	3.914	.566
.229	1.33	4.144	.515
.203	1.19	4.278	.479
.152	1.06	4.415	.431

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TABLE I3

$M_{\infty} = 5.14$	$P_0 = 9.98 \text{ atm}$		
$Re_\theta = 5590$	$T_0 = 380.0 \text{ }^{\circ}\text{K}$		
$C_q = 4.24 \times 10^{-4}$	$T_\infty = 60.48 \text{ }^{\circ}\text{K}$		
$(T_e - T_w)/T_e = 0.1135$	$U_\infty = 801.48 \text{ m/sec}$		
	$T_w/T_\infty = 5.08$		
Y (mm)	M	T/T <sub>∞</sub>	U/U <sub>∞</sub>
11.28	5.14	1.000	1.000
10.64	5.14	1.000	1.000
10.01	5.12	1.007	1.000
9.373	5.07	1.025	.999
8.738	4.97	1.062	.996
8.103	4.82	1.119	.992
7.468	4.63	1.196	.985
6.833	4.41	1.289	.974
6.198	4.175	1.399	.961
5.563	3.93	1.525	.944
4.928	3.66	1.685	.924
4.293	3.42	1.835	.901
3.658	3.165	2.015	.874
3.023	2.935	2.195	.846
2.388	2.71	2.390	.815
1.753	2.48	2.613	.780
1.499	2.39	2.711	.766
1.245	2.28	2.839	.747
.991	2.14	3.019	.723
.864	2.03	3.167	.703
.737	1.93	3.267	.679
.610	1.80	3.437	.649
.483	1.62	3.690	.605
.432	1.54	3.801	.584
.381	1.44	3.942	.556
.330	1.34	4.085	.527
.279	1.23	4.234	.492
.254	1.17	4.315	.473
.229	1.10	4.406	.449
.203	1.02	4.510	.421
.165	.905	4.636	.379
.152	.860	4.684	.362

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TABLE 14

$M_\infty = 5.07$	$P_0 = 10.10$	atm	
$Re_\theta = 6100$	$T_0 = 379.0$	$^{\circ}\text{K}$	
$C_q = 6.18 \times 10^{-4}$	$T_\infty = 61.70$	$^{\circ}\text{K}$	
$(T_e - T_w)/T_e = 0.115$	$U_\infty = 798.5$	m/sec	
$T_w/T_\infty = 4.935$			
$Y$ (mm)	$M$	$T/T_\infty$	$U/U_\infty$
10.617	5.07	1.000	1.000
9.982	5.01	1.024	1.000
9.347	4.87	1.077	.997
8.712	4.70	1.144	.992
8.077	4.50	1.227	.983
7.442	4.28	1.324	.971
6.807	4.05	1.436	.957
6.172	3.807	1.570	.941
5.537	3.57	1.712	.921
4.902	3.33	1.862	.896
4.267	3.09	2.037	.870
3.632	2.887	2.202	.845
2.997	2.67	2.371	.811
2.362	2.46	2.572	.778
1.981	2.30	2.740	.751
1.727	2.23	2.816	.738
1.473	2.11	2.956	.716
1.219	2.00	3.080	.692
.965	1.88	3.241	.667
.711	1.71	3.463	.628
.584	1.58	3.638	.594
.457	1.45	3.815	.559
.330	1.26	4.069	.501
.203	1.03	4.346	.424
.152	.867	4.527	.364

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TABLE 15

$M_\infty$	5.10	$P_0$	10.28	atm
$Re_\theta$	7689	$T_0$	379.3	$^{\circ}\text{K}$
$C_q$	$7.46 \times 10^{-4}$	$T_\infty$	61.18	$^{\circ}\text{K}$
$(T_e - T_w)/T_e$	0.129	$U_\infty$	799.7	m/sec
		$T_w/T_\infty$	4.904	
$Y$ (mm)	$M$	$T/T_\infty$	$U/U_\infty$	
13.18	5.099	1.000	1.000	
10.64	4.914	1.066	.995	
8.103	4.120	1.406	.958	
6.833	3.619	1.690	.923	
5.563	3.204	1.955	.878	
5.309	3.119	2.010	.867	
5.055	3.044	2.072	.859	
4.801	2.957	2.139	.848	
4.547	2.875	2.205	.837	
4.293	2.801	2.274	.828	
4.039	2.704	2.349	.813	
3.785	2.644	2.400	.803	
3.531	2.559	2.479	.790	
3.277	2.476	2.559	.777	
3.023	2.386	2.648	.761	
2.896	2.329	2.708	.752	
2.769	2.292	2.746	.745	
2.642	2.252	2.788	.737	
2.515	2.209	2.833	.729	
2.261	2.162	2.880	.720	
2.007	2.036	3.030	.695	
1.753	1.944	3.189	.681	
1.499	1.850	3.255	.655	
1.245	1.742	3.395	.629	
.991	1.611	3.566	.597	
.737	1.464	3.768	.557	
.483	1.235	4.080	.489	
.356	1.032	4.346	.422	
.229	.874	4.529	.365	
.152	.770	4.626	.325	

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TABLE 16

$M_\infty = 5.20$	$P_0 = 8.39$	atm	
$R_e = 4011$	$T_0 = 383.14$	°K	
$C_q = 0$	$T_\infty = 59.81$	°K	
$(T_e - T_w)/T_e = 0.210$	$U_\infty = 806.4$	m/sec	
	$T_w/T_\infty = 4.615$		
$Y$ (mm)	$M$	$T/T_\infty$	$U/U_\infty$
8.230	5.20	1.000	1.000
7.722	5.19	1.004	1.000
7.214	5.15	1.017	.999
6.960	5.12	1.028	.998
6.325	4.99	1.074	.994
5.944	4.89	1.109	.990
5.690	4.80	1.144	.987
5.055	4.54	1.246	.975
4.928	4.48	1.276	.972
4.420	4.27	1.367	.960
3.785	3.96	1.523	.939
3.150	3.67	1.687	.915
2.388	3.38	1.864	.886
<b>2.134</b>	3.18	2.013	.866
1.880	3.08	2.086	.854
1.372	2.90	2.217	.829
1.118	2.80	2.297	.815
1.067	2.75	2.343	.809
.991	2.69	2.398	.801
.914	2.64	2.445	.794
.864	2.60	2.480	.787
.737	2.47	2.614	.768
.610	2.30	2.810	.741
.483	2.08	3.074	.701
.356	1.75	3.510	.631
.305	1.56	3.744	.589
.254	1.42	3.975	.544
.203	1.27	4.183	.498
.178	1.12	4.380	.451
.152	.999	4.524	.409

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TABLE 17

$M_\infty = 5.16$	$P_0 = 8.283$	atm	
$Re_\theta = 5786$	$T_0 = 387.55$	$^{\circ}\text{K}$	
$C_q = 4.53 \times 10^{-4}$	$T_{\infty} = 61.27$	$^{\circ}\text{K}$	
$(T_e - T_w)/T_e = 0.2428$	$U_\infty = 809.9$	m/sec	
	$T_w/T_\infty = 4.363$		
$Y$ (mm)	$M$	$T/T_\infty$	$U/U_\infty$
10.744	5.16	1.000	1.000
9.474	5.09	1.024	.998
8.839	5.01	1.053	.996
8.204	4.88	1.100	.992
7.569	4.69	1.173	.984
6.934	4.49	1.257	.976
6.680	4.38	1.305	.970
6.299	4.26	1.363	.964
5.664	4.03	1.474	.948
5.029	3.77	1.612	.928
4.394	3.52	1.759	.903
3.759	3.28	1.906	.878
3.124	3.05	2.066	.850
2.489	2.82	2.238	.818
1.854	2.60	2.421	.784
1.219	2.33	2.671	.738
1.092	2.25	2.754	.724
.838	2.09	2.928	.693
.584	1.85	3.213	.644
.457	1.69	3.436	.605
.330	1.46	3.736	.547
.279	1.34	3.890	.512
.229	1.19	4.078	.464
.178	1.072	4.200	.426
.152	.956	4.328	.385

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TABLE 18

$M_\infty = 5.12$	$P_0 = 7.926$	atm	
$Re_\theta = 6736$	$T_0 = 381.65$	$^{\circ}\text{K}$	
$C_q = 6.53 \times 10^{-4}$	$T_{\infty} = 61.64$	$^{\circ}\text{K}$	
$(T_e - T_w)/T_e = 0.2793$	$U_\infty = 802.7$	m/sec	
$T_w/T_{\infty} = 4.105$			
$Y$ (mm)	$M$	$T/T_{\infty}$	$U/U_\infty$
10.719	5.12	1.000	1.000
9.957	5.01	1.036	.996
9.449	4.91	1.074	.994
8.941	4.76	1.131	.989
8.179	4.52	1.230	.979
7.544	4.296	1.331	.968
6.909	4.08	1.435	.955
6.274	3.86	1.551	.939
5.639	3.61	1.694	.918
5.004	3.38	1.835	.894
4.369	3.158	1.985	.869
3.734	2.946	2.132	.840
3.099	2.745	2.280	.810
2.464	2.540	2.445	.776
1.829	2.300	2.655	.732
1.575	2.200	2.750	.713
1.321	2.086	2.866	.690
1.067	1.968	2.990	.665
.813	1.818	3.158	.631
.559	1.602	3.419	.579
.432	1.436	3.629	.534
.305	1.232	3.886	.474
.152	.897	4.272	.362

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TABLE 19

$M_\infty = 5.03$	$P_0 = 7.346$	atm	
$Re_\theta = 7475$	$T_0 = 382.77$	°K	
$C_q = 7.88 \times 10^{-4}$	$T_\infty = 63.18$	°K	
$(T_\theta - T_w) / T_\theta = 0.2825$	$U_\infty = 801.6$	m/sec	
$T_w / T_\infty = 3.973$			
$Y$ (mm)	M	$T / T_\infty$	$U / U_\infty$
11.176	5.02	1.000	1.000
9.906	4.76	1.098	.991
8.636	4.25	1.323	.972
7.366	4.19	1.325	.959
6.096	3.50	1.700	.907
4.826	3.37	1.737	.883
4.191	2.86	2.118	.827
3.556	2.59	2.337	.787
2.921	2.41	2.478	.754
2.286	2.15	2.720	.705
2.032	2.07	2.792	.688
1.778	1.96	2.902	.664
1.524	1.86	3.000	.640
1.270	1.80	3.057	.626
1.016	1.66	3.209	.591
.762	1.57	3.293	.566
.635	1.48	3.391	.543
.508	1.44	3.430	.530
.381	1.34	3.540	.501
.254	1.25	3.641	.474
.229	1.196	3.703	.458
.178	1.114	3.793	.431
.152	1.072	3.838	.418

TABLE 20  
SUMMARY OF BOUNDARY-LAYER PARAMETERS

$M_{\infty}$	$T_w/T_{\infty}$	$C_q$	$\delta$	$\delta^*$	$\theta$	$n$	$Re_{\theta}$	$Re_x$	$C_f$
Set 1 - 29.25 cm Station									
5.05	5.085	0	6.1	2.52	0.242	9.1	3506	4.22 x 10 <sup>6</sup>	9.36 x 10 <sup>-4</sup>
5.06	5.070	1.30 x 10 <sup>-4</sup>	6.4	2.75	0.257	8.3	3739	4.26 x 10 <sup>6</sup>	8.36 x 10 <sup>-4</sup>
5.08	5.043	4.20 x 10 <sup>-4</sup>	7.2	3.30	0.307	7.1	4523	4.31 x 10 <sup>6</sup>	7.60 x 10 <sup>-4</sup>
5.09	5.010	6.16 x 10 <sup>-4</sup>	7.7	3.76	0.347	6.1	5108	4.31 x 10 <sup>6</sup>	6.80 x 10 <sup>-4</sup>
5.09	5.027	7.88 x 10 <sup>-4</sup>	8.2	4.30	0.410	5.2	6178	4.41 x 10 <sup>6</sup>	6.06 x 10 <sup>-4</sup>
Set 2 - 29.25 cm Station									
5.09	4.595	0	6.0	2.33	0.214	9.5	3186	4.35 x 10 <sup>6</sup>	7.83 x 10 <sup>-4</sup>
5.05	4.257	2.52 x 10 <sup>-4</sup>	6.7	2.99	0.272	7.5	3915	4.13 x 10 <sup>6</sup>	7.12 x 10 <sup>-4</sup>
4.94	4.107	6.18 x 10 <sup>-4</sup>	7.6	3.47	0.347	6.7	4747	4.00 x 10 <sup>6</sup>	6.55 x 10 <sup>-4</sup>
4.98	4.219	9.10 x 10 <sup>-4</sup>	8.3	4.57	0.419	4.4	6092	4.25 x 10 <sup>6</sup>	6.45 x 10 <sup>-4</sup>
4.95	3.845	17.80 x 10 <sup>-4</sup>	12.5	8.53	0.722	2.1	10931	4.05 x 10 <sup>6</sup>	2.05 x 10 <sup>-4</sup>
Set 3 - 36.85 cm Station									
5.18	5.204	0	6.9	3.05	0.265	8.5	4000	5.56 x 10 <sup>6</sup>	12.90 x 10 <sup>-4</sup>
5.14	5.038	1.69 x 10 <sup>-4</sup>	7.2	3.39	0.300	7.7	4500	5.51 x 10 <sup>6</sup>	9.21 x 10 <sup>-4</sup>
5.14	5.079	4.12 x 10 <sup>-4</sup>	8.4	4.30	0.367	6.3	5590	5.61 x 10 <sup>6</sup>	9.49 x 10 <sup>-4</sup>
5.07	4.935	5.92 x 10 <sup>-4</sup>	9.0	4.85	0.418	5.3	6100	5.37 x 10 <sup>6</sup>	7.29 x 10 <sup>-4</sup>
5.10	4.904	7.46 x 10 <sup>-4</sup>	9.7	5.78	0.507	4.2	7689	5.59 x 10 <sup>6</sup>	6.63 x 10 <sup>-4</sup>
Set 4 - 36.85 cm Station									
5.20	4.615	0	6.8	2.94	0.268	8.6	4011	5.51 x 10 <sup>6</sup>	10.68 x 10 <sup>-4</sup>
5.16	4.363	4.53 x 10 <sup>-4</sup>	8.7	4.16	0.393	6.3	5786	5.42 x 10 <sup>6</sup>	8.59 x 10 <sup>-4</sup>
5.12	4.105	6.53 x 10 <sup>-4</sup>	9.0	4.87	0.457	5.6	6736	5.44 x 10 <sup>6</sup>	6.91 x 10 <sup>-4</sup>
5.03	3.973	7.88 x 10 <sup>-4</sup>	9.5	5.39	0.532	3.9	7475	5.18 x 10 <sup>6</sup>	5.61 x 10 <sup>-4</sup>

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